



Composite dry sliding bearings – maintenance-free and space-saving



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Maintenance-free and space-saving

Dry sliding bearings of composite material are primarily used for bearing arrangements where heavy loads have to be supported but where rotational or oscillating movements are relatively slow. Because of their good sliding properties and compact design, these plain bearings are particularly suitable for bearing arrangements where

- freedom from maintenance is required,
- there is a risk of lubricant starvation,
- lubricants should not be used or are forbidden, or
- very limited space is available.

Typical application areas for SKF composite dry sliding bearings and examples will be found on **pages 44 to 52**.

Characteristics

The important characteristics of SKF composite dry sliding bearings include

- freedom from maintenance
- no lubrication required (PTFE composite)
- initial lubrication required (POM composite)
- minimum wall thickness, minimum space requirements
- can support heavy loads
- wide operating temperature range
- good sliding properties
- practically without stick-slip
- little wear
- insensitive to edge loads (POM composite)
- no machining required

The assortment

The standard range of SKF composite dry sliding bearings is very comprehensive and meets most needs for this type of product. The range is shown in the product overview (→ **Table 1**) and comprises

- bushings,
- flanged bushings,
- thrust washers, and
- strip.



Maintenance-free and space-saving

Bearing	Series	Bore diameter	Lubrication needed	Permissible temperature range
Plain bushings PTFE composite	PCM .. B	3 – 300 mm	No	-200 to +250 °C
	PCZ .. B	0,125 – 7 inch	No	-200 to +250 °C
POM composite	PCM .. M	8 – 300 mm	Yes, on mounting	-40 to +110 °C short periods +130 °C
	PCZ .. M	0,375 – 4 inch	Yes, on mounting	-40 to +110 °C short periods +130 °C
Flanged bushings PTFE composite	PCMF .. B	6 – 35 mm	No	-200 to +250 °C
Thrust washers PTFE composite	PCMW .. B	15 – 52 mm	No	-200 to +250 °C
POM composite	PCMW .. M	15 – 52 mm	Yes, on mounting	-40 to +110 °C short periods +130 °C
Strip PTFE composite	PCMS .. B	500 × 100 mm Height 1 to 2,5 mm	No	-200 to +250 °C
POM composite	PCMS .. M	500 × 100 mm Height 1 to 2,5 mm	Yes, on mounting	-40 to +110 °C short periods +130 °C

Table 1

Series	Characteristics	Page
PCM .. B	Dry sliding bushings with metric dimensions of triple-layer PTFE composite material (→ page 6). Suitable for slow rotational and oscillating movements under radial load and maintenance-free operation. No lubrication required.	27
PCZ .. B	Dry sliding bushings with inch dimensions of triple-layer PTFE composite material (→ page 6). Suitable for slow rotational and oscillating movements under radial load and maintenance-free operation. No lubrication required.	34
PCM .. M	Dry sliding bushings with metric dimensions of triple-layer POM composite material (→ page 6). Suitable for slow rotational and oscillating movements and also linear movements under radial load and maintenance-free operation but also where there is a risk of errors of alignment, edge loading or moderate contamination. Initial lubrication required.	27
PCZ .. M	Dry sliding bushings with inch dimensions of triple-layer POM composite material (→ page 6). Suitable for slow rotational and oscillating movements and also linear movements under radial load and maintenance-free operation but also where there is a risk of errors of alignment, edge loading or moderate contamination. Initial lubrication required.	34
PCMF .. B	Dry sliding bushings with flange on one side and metric dimensions of triple-layer PTFE composite material (→ page 6). Suitable for slow rotational and oscillating movements under radial and single direction axial loads. For maintenance-free operation. No lubrication required.	38
PCMW .. B	Dry sliding thrust washers with metric dimensions of triple-layer PTFE composite material (→ page 6). Suitable for space-saving arrangements for axial location. For slow rotational and oscillating movements under axial load and maintenance-free operation. No lubrication required.	39
PCMW .. M	Dry sliding thrust washers with metric dimensions of triple-layer POM composite material (→ page 6). Suitable for space-saving arrangements for axial location. For slow rotational and oscillating movements under axial load and maintenance-free operation and where there is a risk of edge loading or moderate contamination. Initial lubrication required.	39
PCMS .. B	Dry sliding strip of triple-layer PTFE composite material (→ page 6). The strip can be bent, pressed, cut etc. to fit the particular application. Suitable for space-saving, maintenance-free arrangements of all types. No lubrication required.	40
PCMS .. M	Dry sliding strip of triple-layer POM composite material (→ page 6). The strip can be bent, pressed, cut etc. to fit the particular application. Suitable for space-saving, maintenance-free arrangements of all types and where there is a risk of moderate contamination or edge loading. Initial lubrication required.	40

Materials

There are two standard types of composite material for SKF dry sliding bearings: PTFE composite and POM composite, which differ in their sliding layers. They are suitable in different bearing applications.

PTFE composite

The PTFE composite material has a copper-plated sheet steel backing on to which a 0,2 to 0,4 mm thick porous layer of tin bronze is sintered (→ **fig 1**). The pores of the sintered layer are filled with a mixture of PTFE (polytetrafluoroethylene) and lead by a rolling process. The sintered bronze layer is covered by a 5 to 30 µm thick running-in layer of the same mixture.

There is an optimum combination of the mechanical properties of the sintered bronze and the good sliding and lubricating properties of the PTFE mixture in PTFE composite bearings. It has good dimensional stability and thermal conductivity.

PTFE composite bearings are identified by designation suffix B, e.g. PCM 101212 B.

POM composite

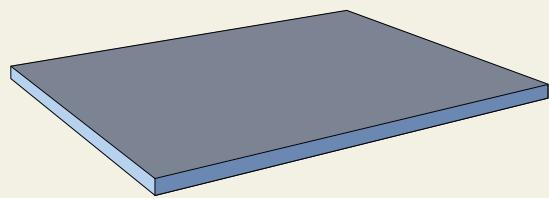
The POM composite material also has a sheet steel backing which is copper plated, and also a 0,2 to 0,4 mm thick layer of sintered tin bronze (→ **fig 2**). The principal characteristic of these bearings is their relatively thick (0,3 mm) covering layer of acetal resin (POM – polyoxymethylene) which is firmly anchored in the sintered bronze layer. The covering layer has pockets to retain lubricating grease.

The thickness of the covering layer makes bearings insensitive to a certain degree of misalignment and the edge loading associated with misalignment.

POM composite bearings are identified by designation suffix M, e.g. PCM 101212 M.

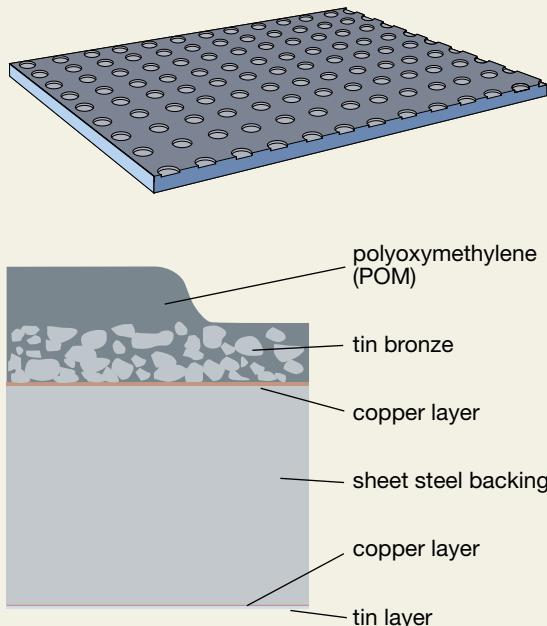
PTFE composite dry sliding material

Fig 1



POM composite dry sliding material

Fig 2



**Summary of characteristics of SKF
PTFE composite and POM composite
dry sliding materials**

Table 2

Characteristics	PTFE composite	POM composite
Composition	steel backing with sintered layer of tin bronze; pore filling and covering layer (5 to 30 µm) of PTFE with lead additives	steel backing with sintered layer of tin bronze; pore filling and covering layer (0,3 mm) of POM
Permissible specific static bearing load (N/mm ²)	250	250
Permissible specific dynamic load (N/mm ²)	80	120
Maximum sliding velocity (m/s)	2	2,5
Operating temperature range (°C)	-200 to +250	-40 to +110 (+130 for brief periods)
Coefficient of friction	0,03 to 0,25	0,02 to 0,20
Stick-slip effect	negligible	negligible
Wear layer thickness (mm)	0,2	0,3
Lubrication	not required	initial lubrication required
Ability to support edge loads (e.g. resulting from misalignment)	fair	good
Ability to carry alternating loads	good	fair
Ability to accommodate linear movement	fair	good
Machining of sliding surface after mounting	calibration	drilling, turning, (reaming)
Recommended housing tolerance for bushings	H7	H7
Recommended shaft tolerance for bushings	f7 (for d ≤ 75 mm) h8 (for d > 75 mm)	h8 h8
Required surface roughness of mating surface (µm)	R _z ≤ 3 R _a ≤ 0,4	R _z ≤ 6 R _a ≤ 0,8
Permissible surface machining of mating surface	ground (drawn)	drawn

Machinability of composite materials

SKF composite dry sliding bearing materials – with the exception of the sliding layer – can be machined using conventional methods.

If bushings are required to have a smaller width than the standard size, this can easily be achieved by turning or parting-off. It is also possible to drill lubrication holes. Care must be taken to see that any burrs are removed, particularly from the sliding surface.

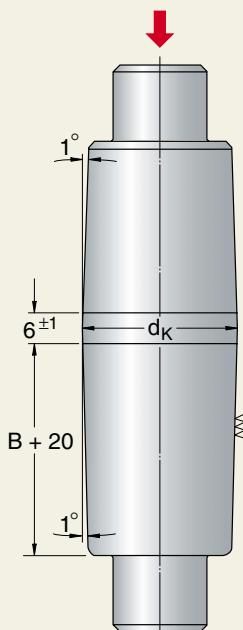
Strip can be bent, coined, pressed, cut, bored or drilled to shape to fit the individual application. When cutting or drilling it is advisable to work from the sliding surface side to avoid creating burrs in the sliding layer.

Any bright metal surfaces produced as a result of machining should be given protection against corrosion.

PTFE composite bushings

These bushings are supplied in a ready-to-mount condition. Any subsequent machining of the bore, i.e. the sliding surface, should only be undertaken in exceptional cases as it will reduce the service life. If necessary, however, the bore should be calibrated using a mandrel (→ fig 3). If a PTFE

Fig 3



Mandrel

d_K = diameter of calibrating mandrel

B = bushing width

Minimum hardness: 50 HRC

Surface roughness: $R_z \approx 1 \mu\text{m}$

composite bushing having an outside diameter D is to be mounted in a steel housing having an outside diameter D_G , guideline values for the requisite diameter d_K of the calibrating mandrel ($= d + \Delta d_K$) will be found in **Diagram 1**. This diagram gives the requisite interference Δd_K of the calibrating mandrel as a function of the desired expansion Δd of the bushing bore for various nominal bushing bore diameters d . The diagram is valid for the ratio $D_G/D = 2$. No values are given for bushings installed in light alloy housings as the influence of design parameters and the housing material is too great. In such cases, the requisite diameter of the calibrating mandrel must be established by trials.

POM composite bushings

Bushings made of the POM composite material are supplied in a ready-to-mount condition but can be machined in the bore, for example, by turning. When machining the bore an R_a value of $2,5 \mu\text{m}$ should be aimed at for the sliding surface. The following machining recommendations have been found to give good results:

- a cutting speed $> 150 \text{ m/min}$,
- a slow feed rate (0,05 to $0,1 \text{ mm/revolution}$),
- a cutting depth $\leq 0,1 \text{ mm}$, and
- a cutting tool of poly-crystalline diamond.

In addition, cooling must be efficient to prevent excessive heating of the plastic with an attendant risk of smearing. The swarf produced must be removed during machining. Both cooling and swarf removal can usually be achieved using compressed air.

Friction

The friction in composite dry sliding bearings is primarily determined by the load, the sliding velocity and the operating temperature. It is also influenced by the roughness of the surface on which the bearing runs, the degree of contamination and the lubrication conditions.

For PTFE composite bearings, the value of the coefficient of friction μ lies between 0,03 and 0,25 depending on the operating conditions. Similar values are found for the POM composite material, but the influence of lubrication is stronger. The lowest values are normally obtained under high specific loads and low sliding velocities, see guideline values for μ in **Diagram 2**. Under particularly unfavourable operating conditions as well as under light loads, the maximum guideline values may even be exceeded. The friction of PTFE composite bearings is increased at temperatures above $+100^\circ\text{C}$.

Stick-slip effects are negligibly small in bearings of both materials.

Chemical properties

The chemical resistance of SKF composite dry sliding bearings is primarily determined by the chemical resistance of the steel backing and the sintered tin bronze layer, as the sliding (covering) layers are chemically resistant to many substances. The covering layer of the PTFE composite material is virtually inert because of its PTFE content, although at elevated temperatures molten alkali metals and free fluorine will attack it. The acetal resin covering layer of the POM composite bearings is largely resistant to organic substances.

At room temperature the sintered tin bronze structure has good resistance to sea water, steam, atmospheric influences, salt solutions and sulphuric acid, but not to oxidising acids or media containing ammonia.

All exposed surfaces of the steel backing are electrolytically tin plated but this gives only limited protection against corrosion in most applications. In cases where the bearings are to be exposed to corrosive media, or where there is a danger of corrosion in the contact between the steel backing and the housing material, the backing can be protected by a nickel, chromium or zinc coating applied electrolytically. Further details can be supplied on request.

Diagram 1

Requisite interference of the mandrel

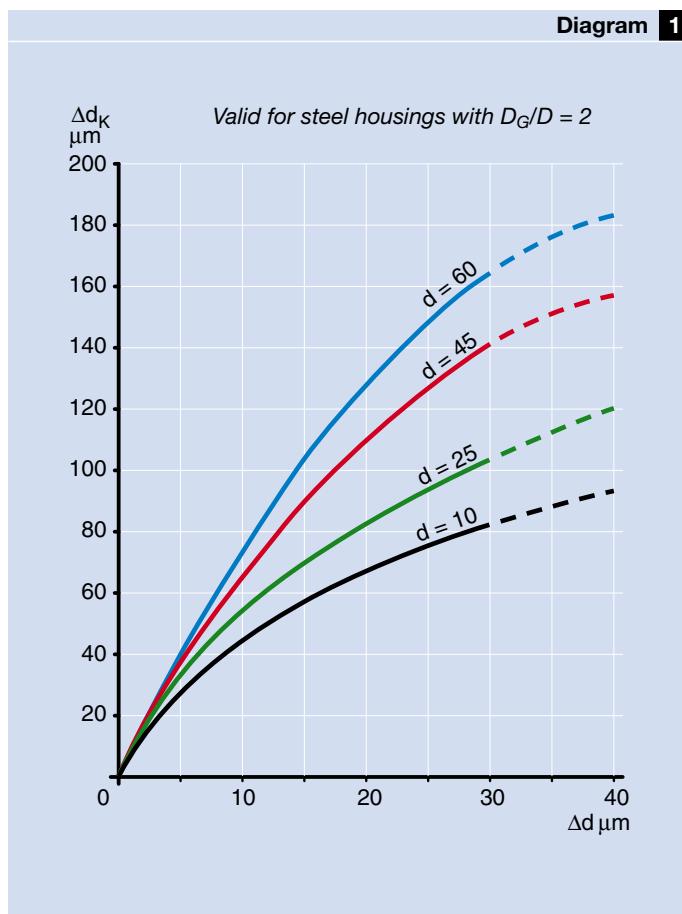
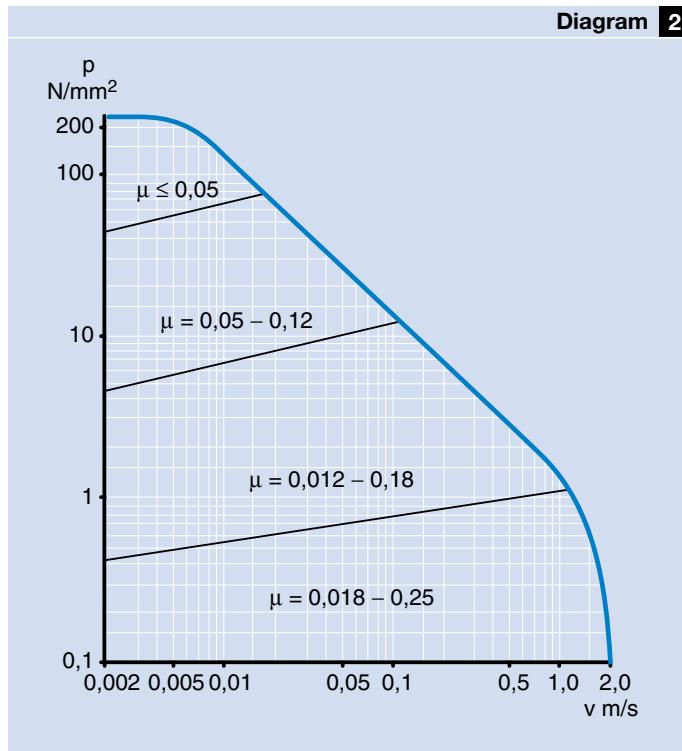


Diagram 2



Guideline values for coefficient of friction of PTFE composite dry sliding bearings

Selection of bearing size

The load carrying ability and wear behaviour of SKF composite dry sliding bearings are governed by the specific conditions pertaining in a particular application. Therefore, any calculation can only provide approximate values. In order to determine the required size of bearing, the load carrying capacity, the applied loads, the service life requirements and operational reliability are all considered. The load carrying capacity is expressed by the basic dynamic load rating C and the basic static load rating C_0 . Values of the load ratings will be found in the product tables.

Basic load ratings

Basic dynamic load ratings

The basic dynamic load rating C is used when calculating dry sliding bearings which are to be dynamically loaded. It is defined as that load, constant in magnitude and direction, under which a given basic rating life (corresponding to a given total distance travelled) can be achieved under constant rotation or oscillating movement at a defined sliding velocity at room temperature. It is assumed in this definition that the load acting on bushings and flanged bushings is purely radial, and the load acting on thrust washers is purely axial and applied at the centre. Dynamic load conditions are essentially oscillating movement or rotation under load, but

also include micro-sliding under variable load (e.g. as a result of vibration) or operation under high-frequency alternating loads. Often a combination of these conditions will be encountered. Whereas oscillating movement or rotation under load usually produces wear, the other conditions may result in fatigue.

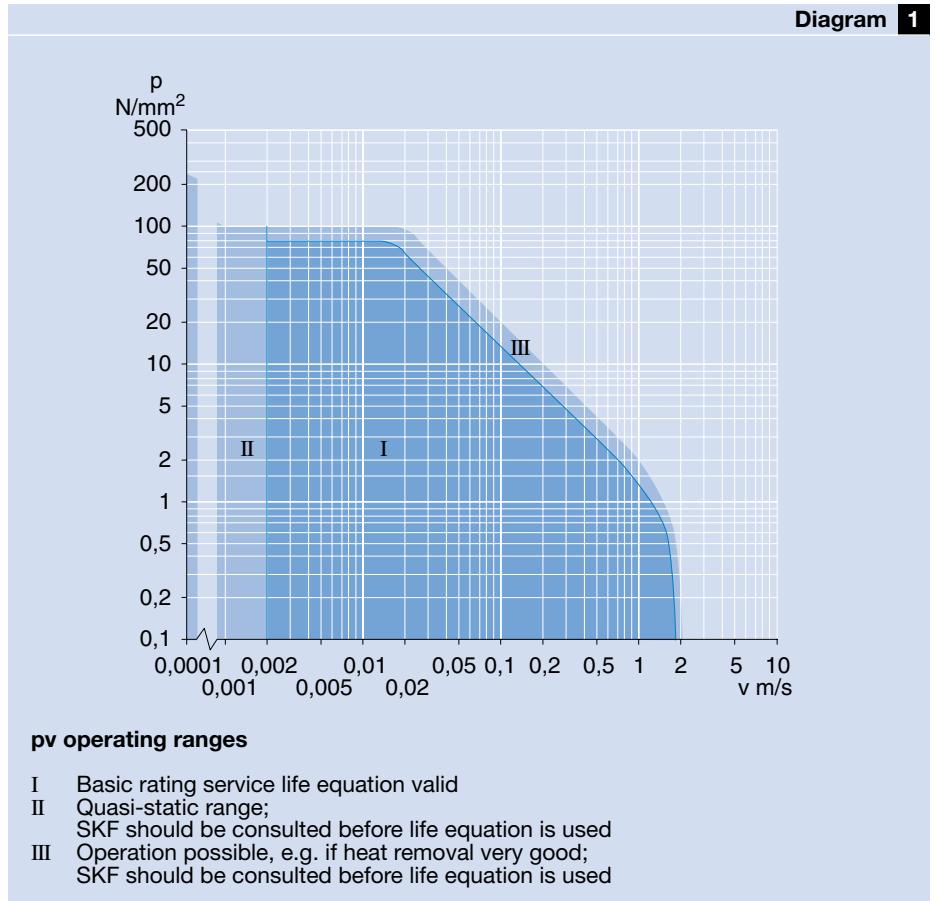
The actual load ratings quoted by various manufacturers depend on the way in which they are defined so that it is not always possible to make direct comparisons between them.

Basic static load ratings

The basic static load rating C_0 is defined as the maximum load which an SKF composite bushing, flanged

bushing or thrust washer can support when stationary at room temperature without permanent deformation of the sliding layer being produced which would jeopardise its performance. It is assumed here that the components surrounding the bearing prevent its deformation. At elevated temperatures it is necessary to modify the basic load ratings of the various materials by multiplying the C_0 value by the temperature factor c_3 , which is also valid for dynamically loaded bearings (→ **Diagram 5**). The permissible operating temperature range should also be considered (→ **Table 2**), page 7).

Diagram 1



**pv operating range
for PTFE composite
dry sliding bearings**

Service life

The service life of a dry sliding bearing is expressed as a number of oscillations or revolutions, or in operating hours. It depends on the clearance increase occurring under boundary or dry lubrication conditions because of the continuing wear of the sliding contact surfaces, plastic deformation of the sliding layer or fatigue. Depending on the application and sliding layer various degrees of wear or increases in friction may be acceptable. This means that even under apparently similar operating conditions, the service life achieved in practice will differ, simply because the requirements placed on the bearing differ.

In contrast, the lives actually achieved by seemingly identical bearings under identical operating conditions for identical demands are not the same. This scatter of results has been

found both in laboratory endurance tests as well as in field tests. Obviously the actual lives will also be affected by the actual operating conditions – not only the magnitude and type of load but also many other influences which are difficult or even impossible to quantify. These include contamination, corrosion, high frequency load and movement cycles and shock loads.

However, the basic rating service life is a guideline value which is attained or exceeded by the majority of bearings under the test conditions.

Requisite bearing size

The type and mode of action of the load, the expected operating temperature, lubrication and maintenance requirements etc. all influence the choice of bearing type and design.

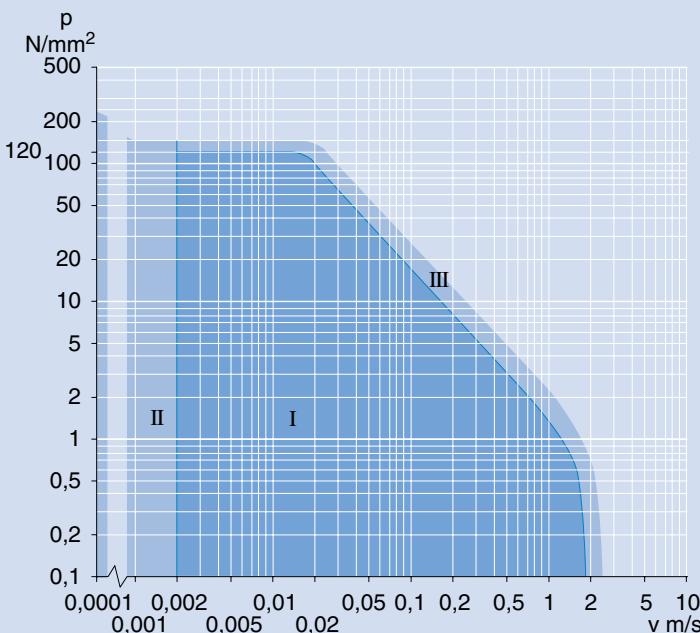
To determine the requisite size of composite dry sliding bearing to be used, it is necessary to know the basic rating service life which is required for, or appropriate to, a given application. This depends on the type of machine, the duration of operation, the operating conditions and the degree of operational reliability required.

pv operating range

When selecting a suitable size of dry sliding bearing **Diagrams 1** and **2** can be used to check whether a proposed bearing can be used under the given load and at the given sliding velocity. The data required – p (specific bearing load) and v (sliding velocity) – can be calculated using the equations overleaf. If it is found that the operating data lie within range I of the pv diagram, the basic rating service life of the bearing can be determined using the equation given in the following section. If the data lie within range II or III of the diagram, either SKF should be contacted, or pre-trials made to see whether the bearing can be used. Alternatively, a different bearing should be chosen so that the values for p and v fall within the range I.

For flanged bushings it is necessary to check the suitability of the bushing and flange separately.

Diagram **2**



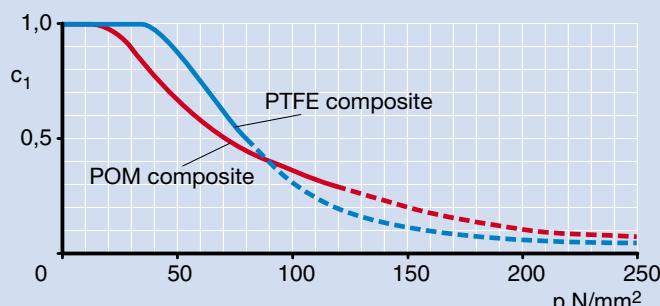
pv operating ranges

- I Basic rating service life equation valid
- II Quasi-static range;
SKF should be consulted before life equation is used
- III Operation possible, e.g. if heat removal very good;
SKF should be consulted before life equation is used

**pv operating range
for POM composite
dry sliding bearings**

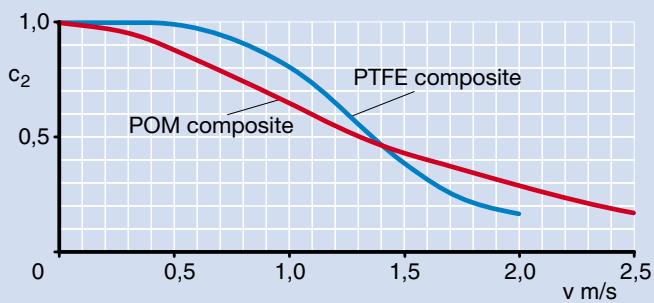
Selection of bearing size

Diagram 3



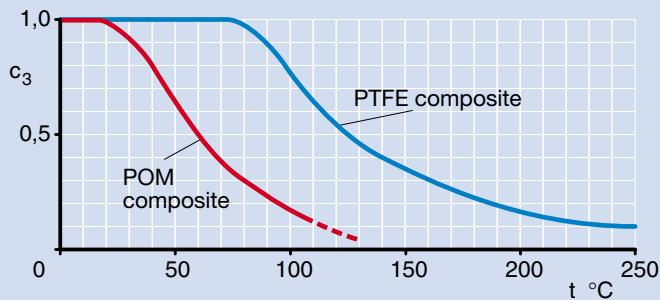
Load factor c_1

Diagram 4



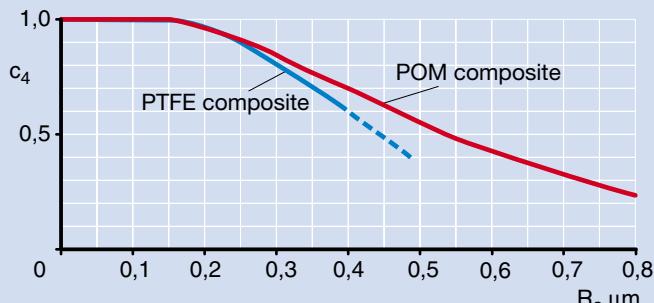
Speed factor c_2

Diagram 5



Temperature factor c_3

Diagram 6



Surface roughness factor c_4

Determination of specific bearing load

The specific bearing load can be determined from

$$p = K \frac{F}{C}$$

where

- p = specific bearing load, N/mm^2
- F = dynamic bearing load, N
- C = basic dynamic load rating, N
- K = specific load factor, N/mm^2
 - = 80 for PTFE composite material
 - = 120 for POM composite material

For flanged bushings it is necessary to calculate the specific load of the bushing and flange separately. When calculating the specific bearing load for the flange the axial basic dynamic load rating C_a should be used instead of C in the above equation. Values of C_a will be found in the product tables.

Determination of sliding velocity

The sliding velocity for SKF composite bushings and thrust washers can be obtained from

$$v = 5,82 \times 10^{-7} d \beta f$$

where

- v = sliding velocity, m/s
- d = bore diameter of bushings and flanged bushings, mm
 - = mean diameter of flange of flanged bushings = $0,5(d + D_1)$, mm
 - = mean diameter of thrust washers = $0,5(d + D)$, mm (= dimension J in product table)
- f = frequency of oscillation, min^{-1} , or rotational speed, r/min
- β = half the angle of oscillation, degrees (\rightarrow fig 1)
- A complete oscillation (from point 0 to point 4) = 4β . For rotation, use $\beta = 90^\circ$.

Calculation of service life

Many factors influence the life of a dry sliding bearing, e.g. load, sliding velocity, operating temperature, surface roughness of the surface on which the dry sliding layer runs etc. Any calculation is therefore only approximate.

The values obtained using the equations given below for the basic rating service life are attained by the majority of bearings and are often exceeded. This has been confirmed by rig tests and field experience.

The basic rating service life for SKF PTFE composite and POM composite dry sliding bearings can be calculated from

$$G_h = c_1 c_2 c_3 c_4 c_5 \frac{K_M}{(pv)^n}$$

where

G_h = basic rating service life, operating hours

c_1 = load factor (**Diagram 3**)

c_2 = speed factor (**Diagram 4**)

c_3 = temperature factor
(**Diagram 5**)

c_4 = surface roughness factor
(**Diagram 6**)

c_5 = factor for the type of load
= 1 for point load (i.e. the loaded zone is always at the same position on the bearing circumference)
= 1,5 for rotating load (i.e. the loaded zone moves round the circumference of the bearing)

K_M = factor depending on material and bearing type
= 480 for PTFE composite bushings

= 300 for PTFE composite thrust washers

= 1 900 for POM composite bushings and thrust washers

p = specific bearing load, N/mm²

v = sliding velocity, m/s

n = an exponent

= 1 for PTFE composite bushings and thrust washers

= 1 for $pv \leq 1$ for POM composite bushings and thrust washers

= 3 for $pv > 1$ for POM composite bushings and thrust washers

If loads are very light and/or sliding velocities very low and the value of the product pv

- for PTFE composite bearings is less than the limiting value of 0,025 then the limiting value $pv = 0,025$ should be used for the life calculations.
- for POM composite bearings is less than the limiting value of 0,1 then the limiting value $pv = 0,1$ should be used for the life calculations.

Calculation example

The suspension of a rail vehicle is to be equipped with composite dry sliding bearings at the linkage position of the springs; in this case bushings arranged in pairs are to be used.

Design data:

Pin diameter: $d = 30$ mm

Surface roughness of pin: $R_a = 0,4 \mu\text{m}$

Operating data:

Radial load at the linkage point:

$F_r = 18\ 750$ N

Half angle of oscillation: $\beta = 1^\circ$

(**fig 1**)

Frequency of oscillation: $f = 180 \text{ min}^{-1}$

Operating temperature: $t = 30^\circ\text{C}$

Based on the design characteristics, PTFE composite bushing PCM 303420 B having a basic dynamic load rating $C = 46\ 500$ N is chosen. It is necessary to check that the bushing can be used under the given operating conditions and then to calculate the basic rating service life.

As a first check that the bearing size is suitable (**Diagram 1**) the specific bearing load p is calculated using

$$p = K \frac{F}{C} = 80 \times \frac{18\ 750}{2 \times 46\ 500} \approx 16 \text{ N/mm}^2$$

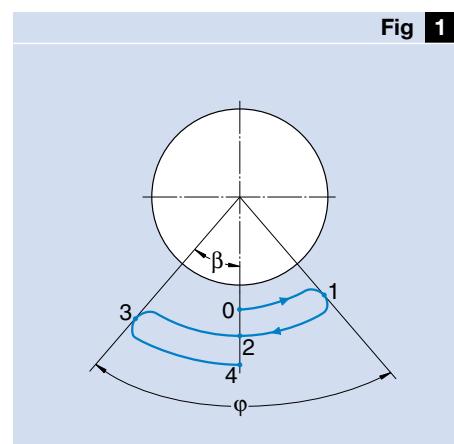
(with the specific load factor $K = 80$ for PTFE composite).

The sliding velocity is calculated using

$$v = 5,82 \times 10^{-7} d \beta f$$

$$= 5,82 \times 10^{-7} \times 30 \times 1 \times 180$$

$$= 0,0031 \text{ m/s}$$



Angle of oscillation

$$\varphi = \text{angle of oscillation} = 2 \beta$$

A complete oscillation = 4β
(from point 0 to point 4)

These values are within the permissible range I of the pv diagram for PTFE composite bearings. Furthermore

the load factor $c_1 = 1$ (**Diagram 3**), the speed factor $c_2 = 1$ (**Diagram 4**), the temperature factor $c_3 = 1$ (**Diagram 5**), the surface roughness factor $c_4 = 0,6$ (**Diagram 6**), and the factor for the type of load $c_5 = 1,5$ as the load is rotating.

The value of K_M for PTFE composite bushings = 480. Thus using the basic rating service life equation for SKF composite dry sliding bearings

$$G_h = c_1 c_2 c_3 c_4 c_5 \frac{K_M}{(pv)^n}$$

$$G_h = 1 \times 1 \times 1 \times 0,6 \times 1,5 \times \frac{480}{16 \times 0,0031}$$

$$G_h = 8\ 700 \text{ operating hours}$$

Application of bearings

Material and surface finish of counterfaces

The factors which are most important to consider when selecting the material and surface finish of the counterface (the surface on which the bearing slides) are the loading conditions (load, angle of oscillation, type of movement etc.) and the environmental influences.

Where there is a risk of corrosion, the counterface must be sufficiently resistant. Corrosion scars in the counterface and the products of corrosion (particulate contamination) increase the surface roughness or are abrasive, thus increasing wear. In such cases the use of stainless steel or a surface treatment such as hard chromium or nickel plating or electrolytic oxidation should be considered.

For PTFE composite and POM composite dry sliding bearing arrangements soft carbon steels having a ground surface are usually adequate for the counterface. The surface roughness R_a (to DIN 4768:1990) should not exceed 0,4 µm for PTFE composite bearings and 0,8 µm for POM composite bearings. The corresponding R_z values are 3 and 6 µm, respectively. For more demanding applications the use of hardened shafts is recommended. These should have a surface hardness of at least 50 HRC. Alternatively, hard chromium or nickel plating or some other form of surface treatment should be considered. In all cases R_a should not be greater than 0,3 µm ($R_z \leq 2 \mu\text{m}$). The better the surface finish, the better the running properties and the less the wear (→ **Surface roughness factor c₄, Diagram 6, page 12**).

Shaft and housing tolerances

It is recommended that the seating in the housing bore is machined to tolerance H6 for PTFE composite and POM composite dry sliding bushings (including flanged bushings) having a bore diameter up to and including 4 mm, and to tolerance H7 for larger bearings. If this is done, after mounting, the bore diameter of the bushing and the clearance in the bearing arrangement will lie within the limits quoted in **Tables 1** and **3** for metric sizes of PTFE composite and POM composite dry sliding bushings, respectively, provided the shaft seating also has the recommended tolerance.

The shaft and housing limits for inch-sized bushings are given in **Tables 2** and **4**, respectively, together with the corresponding limits for the bushing bore diameter after mounting and the operating clearance.

The values quoted are for the operating clearance at room temperature. If the operating temperature is higher than this it is expected that the operating clearance of

- PTFE composite bushings will be reduced by 0,0016 mm and
- POM composite bushings will be reduced by 0,005 mm

for every 20 °C temperature increase. The actual operating clearance can be increased or decreased within the recommended limits by matching shaft and housing bores having appropriate values within the specified limits.

If in certain applications very easy running is required, for example, or if the bearing is only lightly loaded, it is recommended that maximum values for the operating clearance should be aimed for.

The recommended tolerances and guideline limits quoted in the tables are valid for steel and cast iron housings.

Where light alloy housings are used, a greater degree of interference may be required because of the different thermal expansion characteristics. There is otherwise a risk that the greater thermal expansion of the housing would no longer provide radial location for the bushing and that the operating clearance would be too large.

If it is not possible to adopt a greater interference for mounting reasons, or because of the force required to press the bushing into the housing bore, it is possible to use an adhesive to retain the bushing in position. In special cases it may be necessary, by selecting a suitable tolerance for the shaft, to prevent an inadmissible increase in operating clearance.

Application of bearings

PTFE composite bushings (metric sizes)
Shaft and housing tolerances, bearing clearance

Continuation of Table 1

Bushing dimensions				Diameter limits		Housing bore		Bore diameter of mounted bushing		Operating clearance or preload (-)	
Bore diameter	Outside diameter	Wall thickness	Shaft (h8)	max	min	max	min	max	min	min	max
d	D	max	min	max	min	max	min	max	min	min	max
mm											
160	165	2,465	2,415	160,000	159,937	165,040	165,000	160,210	160,070	70	273
180	185	2,465	2,415	180,000	179,937	185,046	185,000	180,216	180,070	70	279
200	205	2,465	2,415	200,000	199,928	205,046	205,000	200,216	200,070	70	288
210	215	2,465	2,415	210,000	209,928	215,046	215,000	210,216	210,070	70	288
220	225	2,465	2,415	220,000	219,928	225,046	225,000	220,216	220,070	70	288
250	255	2,465	2,415	250,000	249,928	255,052	255,000	250,222	250,070	70	294
300	305	2,465	2,415	300,000	299,919	305,052	305,000	300,222	300,070	70	303

Application of bearings

PTFE composite bushings (inch sizes)
Shaft and housing tolerances, bearing clearance

Continuation of Table 2

Bushing dimensions				Diameter limits Shaft		Housing bore		Bore diameter of mounted bushing		Operating clearance or preload (-)	
Bore diameter	Outside diameter	Wall thickness		max	min	max	min	max	min	max	min
inch/mm		inch/mm						μinch/μm			
1,875 47,625	2,0625 52,388	0,0939 2,386	0,0921 2,340	1,8737 47,592	1,8721 47,551	2,0633 52,408	2,0621 52,377	1,8791 47,728	1,8742 47,605	0,51 13	6,97 177
2 50,8	2,1875 55,563	0,0939 2,386	0,0921 2,340	1,9987 50,767	1,9969 50,721	2,1883 55,583	2,1871 55,552	2,0041 50,903	1,9992 50,780	0,51 13	7,17 182
2,25 57,15	2,4375 61,913	0,0925 2,35	0,0906 2,30	2,2507 57,168	2,2489 57,122	2,4377 61,918	2,4365 61,887	2,2566 57,318	2,2515 57,187	0,75 19	7,72 196
2,5 63,5	2,6875 68,263	0,0925 2,35	0,0906 2,30	2,5011 63,528	2,4993 63,482	2,6881 68,278	2,6869 68,247	2,5070 63,678	2,5019 63,547	0,75 19	7,72 196
2,75 69,85	2,9375 74,613	0,0925 2,35	0,0906 2,30	2,7500 69,850	2,7482 69,804	2,9370 74,600	2,9358 74,569	2,7559 70,000	2,7507 69,869	0,75 19	7,72 196
3 76,2	3,1875 80,963	0,0925 2,35	0,0906 2,30	3,0000 76,200	2,9982 76,154	3,1872 80,955	3,1858 80,919	3,0061 76,355	3,0007 76,219	0,75 19	7,91 201
3,5 88,9	3,6875 93,663	0,0925 2,35	0,0906 2,30	3,5000 88,900	3,4978 88,844	3,6872 93,655	3,6858 93,615	3,5061 89,055	3,5007 88,919	0,75 19	8,31 211
4 101,6	4,1875 106,363	0,0925 2,35	0,0906 2,30	4,0000 101,600	3,9978 101,544	4,1872 106,355	4,1858 106,319	4,0061 101,755	4,0007 101,619	0,75 19	8,31 211
5 127	5,1875 131,763	0,0915 2,325	0,0896 2,275	4,9986 126,964	4,9961 126,901	5,186 131,724	5,1844 131,684	5,0067 127,174	5,0013 127,034	2,6 70	10,7 273
6 152,4	6,1875 157,163	0,0915 2,325	0,0896 2,275	6,0000 152,400	5,9975 152,337	6,1874 157,160	6,1858 157,119	6,0083 152,610	6,0027 152,469	2,72 69	10,7 273
7 177,8	7,1875 182,563	0,0915 2,325	0,0896 2,275	6,9954 177,683	6,9929 177,620	7,1830 182,448	7,1812 182,403	7,0039 177,888	6,9981 177,753	2,76 70	10,9 278

POM composite bushings (inch sizes)
Shaft and housing tolerances, bearing clearance

Continuation of Table 4

Bushing dimensions		Wall thickness		Diameter limits Shaft		Housing bore		Bore diameter of mounted bushing		Operating clearance	
Bore diameter	Outside diameter	max	min	max	min	max	min	max	min	min	max
inch/mm		inch/mm									μ inch/ μ m
3 76,2	3,1875 80,963	0,0991 2,517	0,0965 2,451	2,9849 75,817	2,9831 75,771	3,1889 80,998	3,1875 80,963	2,9959 76,096	2,9893 75,929	4,41 112	12,8 325
3,5 88,9	3,6875 93,663	0,0991 2,517	0,0965 2,451	3,4844 88,504	3,4822 88,448	3,6889 93,698	3,6875 93,663	3,4959 88,796	3,4893 88,629	4,92 125	13,7 348
4 101,6	4,1875 106,363	0,0991 2,517	0,0965 2,451	3,9839 101,191	3,9817 101,135	4,1889 106,398	4,1875 106,363	3,9959 101,496	3,9893 101,329	5,43 138	14,2 361

Design of associated components

Bushings

The surface of the shaft on which the bushing runs, i.e. the counterface, should always be wider than the actual bushing – particularly where axial displacement of the shaft relative to the housing may occur as a result of changes in shaft length – in order to prevent step formation in the sliding surface.

To ease mounting, shaft ends and housing bores should have a lead-in chamfer with an angle of 10 to 15° (→ fig 1). It is then easier to press the bushings into the housing bore and to insert the shaft into the bushing bore without the risk of damaging the sliding surface.

The housing shoulders intended for axial location of the bushing should have a diameter which is equal to or greater than $d + 0,8$ mm.

Where PTFE composite bushings operate without lubricant it is especially important to accurately align bearing positions. If misalignment between the positions cannot be avoided, it is necessary to take steps at the design stage to prevent inadmissibly high edge stresses from occurring. For

example, the housing bore seating should be relieved at both sides, or a wider bushing should be used so that it extends beyond the housing bore seating at both sides (→ fig 2).

If errors of alignment have to be compensated for and the operating conditions permit the use of POM composite, then bushings of this material should be chosen. The covering layer of this material can be machined to a minimum degree after the bushing has been mounted in a housing bore.

Flanged bushings and thrust washers

For shafts which not only need radial support, but also require axial location, flanged bushings or a combination of bushing and thrust washer (→ fig 3) can be used, depending on the magnitude of the axial load. The use of flanged bushings or thrust washers is advantageous even where axial loads are small, particularly where suitable surfaces are not available to take the thrust, either because the material or its finish is unsuitable.

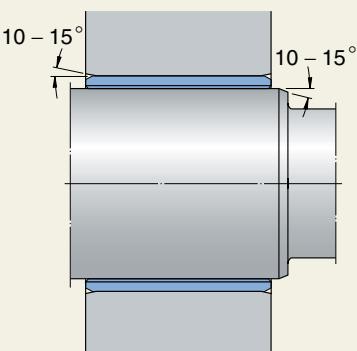
It should be remembered that the counterface should completely cover the sliding surface of the thrust washer and the flange of a flanged bushing (→ fig 4). For bearing arrangements where flanged bushings are used the

transition from housing bore to abutment should be chamfered so that it does not contact the bushing at the transition to the flange (→ fig 5).

Thrust washers are generally located radially in a turned recess in the housing (→ fig 3) and secured by a dowel pin or grub screw to prevent them from rotating. The appropriate dimensions for this type of location are given in the product tables. If a recess cannot be provided in the housing for some reason, the thrust washer can be attached to the housing by two pins or screws (→ fig 6) or by glueing. The heads of the pins or screws must be recessed to at least 0,3 mm below the sliding surface and the entire surface of the thrust washer must be supported.

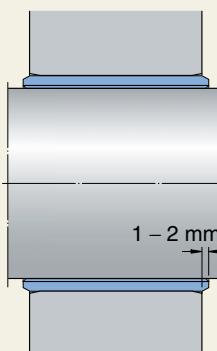
Lead-in chamfers for housing bores and shaft ends

Fig 1



Bushing extending beyond the bearing seating in the housing at both sides to prevent inadmissibly high edge stresses

Fig 2



Combination of bushing and thrust washer

Fig 3

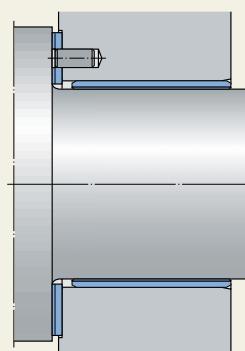
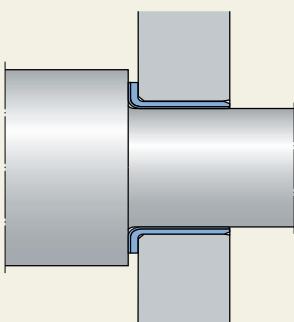
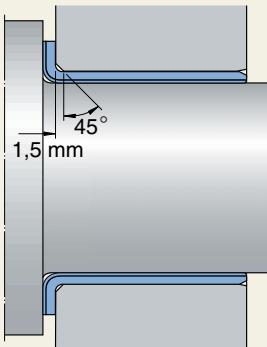
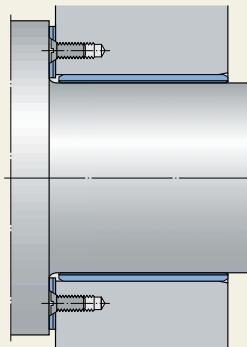


Fig 4**Fig 5****Fig 6**

The mating surface should cover the entire surface of the flange

The transition between housing bore and support surface must be sufficiently large

Thrust washer secured by two grub screws

Seals

The service life of composite dry sliding bearings is decisively influenced by the seals used. When selecting suitable seals it is necessary to consider, for example, the design, the available space and the justifiable expense.

Composite dry sliding bearings, in particular those of the POM composite, are able to embed contaminant particles and are thus relatively insensitive to contamination. They generally require no special protection against normal airborne dirt. If, however, the bearing position is subjected to heavier contamination it should be sealed off from the outside. Simple and effi-

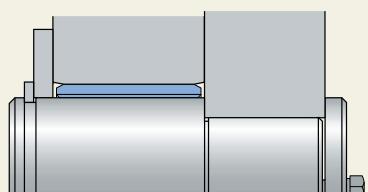
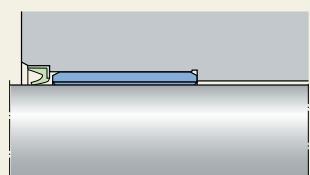
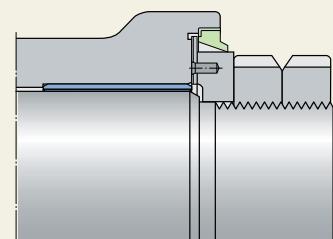
cient sealing can be obtained if adjacent components can also serve as seals (→ fig 7). Radial shaft seals with low cross section, e.g. of the G design, provide adequate protection for composite dry sliding bearings in normal cases (→ fig 8). If the demands placed on the sealing arrangement are high, it may be necessary to resort to special seals of rubber, plastic or similar materials (→ fig 9).

Under very rough conditions, particularly where sand or clay contaminants occur, rubber or plastic seals usually have a very short life. Good “sealing” will be achieved in such cases by periodic relubrication, if the operating conditions permit.

Adjacent components serve as seals

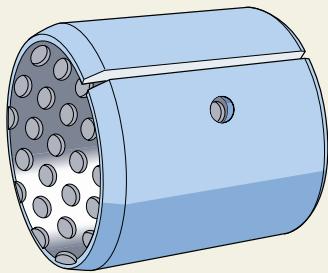
A shaft seal with low cross sectional height can be used

Sealing with a wiper-scaper seal of nitrile rubber

Fig 7**Fig 8****Fig 9**

Lubrication

Fig 1



POM composite bushing with lubrication hole

PTFE composite bearings

PTFE composite bearings have good dry sliding characteristics and do not require lubrication. The presence or continuous supply of oil or other fluid may be advantageous, however. Even fluids not normally associated with lubrication, such as water, kerosene or paraffin, may be used.

Lubrication improves the removal of heat from the bearing position and the formation of a hydrodynamic lubricating film has a very favourable effect on the wear behaviour of the bearing and considerably extends bearing life.

If periodic relubrication with grease is used to enhance sealing or to protect the counterface against corrosion, bearing life will also be extended. A single initial application of grease when mounting PTFE composite bearings may, however, have a negative influence on bearing life, as the grease will form a paste-like mixture with the wear particles produced during running in. This "paste" will increase bearing wear.

Age-resistant lithium base greases are preferred for operating temperatures up to 80 °C, while at higher temperatures, silicone greases should be used. Greases containing solid lubricants such as molybdenum disulphide are totally unsuitable.

POM composite bearings

POM composite dry sliding bearings require an initial application of grease on mounting. Relubrication is not required but the presence or constant supply of lubricating fluid or periodic grease relubrication serve to extend bearing life considerably.

For this reason POM composite bushings having a bore diameter of 10 mm and above and a width of 12 mm or more are supplied as standard with a lubrication hole (→ fig 1). The hole dimensions and position are in accordance with DIN 1434-3:1983. The same greases as those recommended above can be used.

Mounting

Skill and care in mounting are prerequisites for the successful performance of bearings and the avoidance of premature wear.

The counterface (shaft seating) and other components of the bearing arrangement should be carefully cleaned and deburred before mounting is begun. Unmachined surfaces in cast iron housings must be free of sand. The condition of the shaft should be carefully checked so that there are no sharp edges or burrs or surface defects which would damage the sliding surface of the bushings as they are mounted.

A mounting dolly is the most suitable tool for mounting PTFE composite and POM composite dry sliding

bushings and flanged bushings (→ fig 1). An O-section rubber ring placed on the dolly is a simple means of retaining the bushing in position. The use of a mounting ring (→ fig 2) is recommended for larger bushings as it aligns and centres the bushing so that it will not tilt or skew when being pressed in. A light oiling or greasing of the seating in the housing makes mounting easier. When mounting larger bushings it has been found that using a solid lubricant paste on the seating is beneficial in reducing the risk of fretting corrosion and also reducing the force required for mounting.

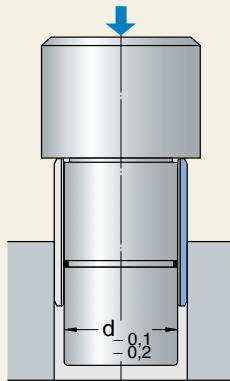
Composite dry sliding bushings and flanged bushings which are to be heavily loaded in operation should be

mounted so that the butt joint is at 90° to the loaded zone in operation (→ fig 3), otherwise life will be reduced. When mounting thrust washers care should be taken that they are correctly positioned, i.e. the steel backing should abut the housing wall.

If composite dry sliding bearings are to be located in the housing using adhesive, it should be remembered that the adhesive used should be suitable for the expected operating temperature and should have suitable expansion, ageing resistance, strength and curing properties. If no operational experience is available, it is recommended that the adhesive manufacturer be contacted for advice. When applying adhesive care must be taken to see that no adhesive reaches the sliding surface.

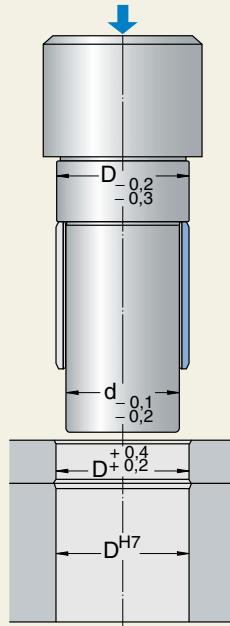
Mounting with a mandrel

Fig 1



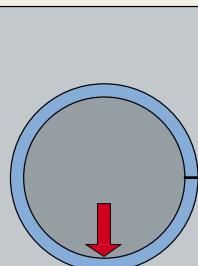
Mounting with a mandrel and mounting ring

Fig 2



The main direction of the load must be at 90° to the plane of the butt joint

Fig 3



Bearing data - general

Dimensions

The dimensions of the metric sizes of PTFE composite and POM composite dry sliding bushings in the bore diameter range 4 to 160 mm, inclusive, are in accordance, with a few exceptions, with those specified in ISO 3547-1976 and DIN 1494/1:1983.

The dimensions of the inch-size bushings, the flanged bushings and the thrust washers have not been standardised.

Tolerances

Bushings: The tolerances for the outside diameter of the metric sizes of SKF PTFE composite and POM composite dry sliding bushings correspond to DIN 1494/1:1983. To check the values, the procedure given in DIN 1494/2:1983 should be used. For all sizes, the tolerances for the width B are a uniform $\pm 0,25$ mm.

Flanged bushings: For all sizes, when mounted, the tolerances are a uniform $\pm 0,5$ mm for the flange diameter D_1 and for the width B_1 $+0,05/-0,20$ mm.

Thrust washers: The tolerances for the diameters are given in the product table. The tolerances for the height are

- 0/-0,05 mm for PTFE composite thrust washers
- 0/-0,10 mm for POM composite thrust washers.

Strip: The tolerances for the height are

- 0/-0,05 mm for PTFE composite strip
- 0/-0,10 mm for POM composite strip

Operating clearance

The operating clearance of bushings depends on the recommended shaft and housing tolerances. Guideline values for bushings in metric sizes are

given in **Tables 1** and **3** and for inch-sized bushings in **Tables 2** and **4**.

Excessive clearance may have a negative influence on the service life of PTFE composite bushings if they are not lubricated.

Permissible operating temperature range

PTFE composite dry sliding bearings can be used at temperatures between -200 and $+250$ °C.

The operating temperature range for POM composite bearings is -40 to $+110$ °C, although brief periods of operation at $+130$ °C are permissible.

The service life of SKF composite dry sliding bearings will be shortened when operating at temperatures above a given value. This is taken into account when calculating the basic rating service life by the temperature factor c_3 (→ **Diagram 5**, page 12).

Running-in

During a short running-in phase there will be some transfer of material from the covering layer of bearings made from PTFE composite to the counterface. After this transfer has taken place, the characteristic low friction and wear properties of these bearings will be obtained.

Electrical properties

Bearings made from POM composite, because of their acetal resin covering layer, may act as electrical insulators when new. To avoid the build-up of static electricity, components at risk should be earthed.

Product designations

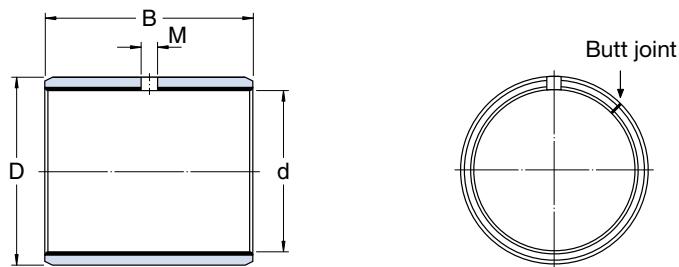
SKF metric composite dry sliding bearings are identified by designations made up of a prefix PCM which may have a fourth letter indicating the type of bearing (e.g. PCMW for a thrust

washer) followed by 6 to 9 figures giving the dimensions (d, D, B/H) in millimetres uncoded. The small bushings carrying the additional suffix /VB055 are an exception to this: the outside diameter is 0,5 mm larger than indicated in the designation. The actual material used is identified by a suffix: B for PTFE composite and M for POM composite. For example, PCM 081008 M is a POM composite bushing with d = 8 mm, D = 10 mm and B = 8 mm.

The inch-size bearings have similar designations, but in this case the prefix is PCZ and the size (d, B) is shown in 1/16ths of an inch, e.g. PCZ 1208 B is a PTFE composite bushing with d = 12/16" = 3/4" and B = 8/16" = 1/2".

Composite bushings with metric dimensions

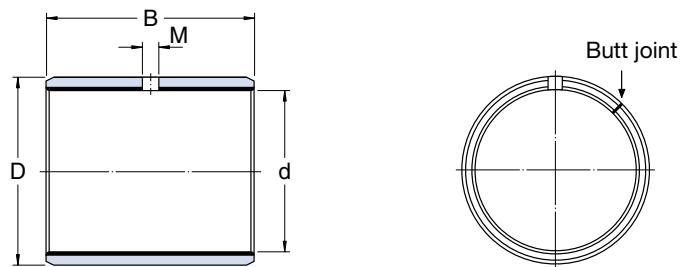
d 3 – 12 mm



Dimensions				Basic load ratings	Mass	Designations	
d	D	B	M	dynamic C	static C ₀	PTFE composite	POM composite
mm				N	g	–	
3	4,5	3	–	720	2 240	0,2	PCM 030403 B/VB055
	4,5	5	–	1 200	3 750	0,3	PCM 030405 B/VB055
	4,5	6	–	1 430	4 500	0,4	PCM 030406 B/VB055
4	5,5	3	–	965	3 000	0,2	PCM 040503 B/VB055
	5,5	4	–	1 270	4 000	0,3	PCM 040504 B/VB055
	5,5	6	–	1 930	6 000	0,6	PCM 040506 B/VB055
	5,5	10	–	3 200	10 000	0,8	PCM 040510 B/VB055
5	7	5	–	2 000	6 200	0,7	PCM 050705 B
	7	8	–	3 200	10 000	1,1	PCM 050708 B
	7	10	–	4 000	12 500	1,4	PCM 050710 B
6	8	6	–	2 900	9 000	1,0	PCM 060806 B
	8	8	–	3 800	12 000	1,3	PCM 060808 B
	8	10	–	4 800	15 000	1,6	PCM 060810 B
7	9	10	–	5 600	17 600	1,8	PCM 070910 B
8	10	6	–	3 800	12 000	1,2	PCM 081006 B
	10	8	–	5 100	16 000	1,7	PCM 081008 B
	10	8	–	7 650	16 000	1,3	–
	10	10	–	6 400	20 000	2,1	PCM 081010 B
	10	10	–	9 650	20 000	1,6	–
	10	12	–	7 650	24 000	2,5	PCM 081012 B
	10	12	–	11 600	24 000	1,9	–
10	12	8	–	6 400	20 000	2,0	PCM 101208 B
	12	10	–	8 000	25 000	2,5	PCM 101210 B
	12	10	–	12 000	25 000	1,9	–
	12	12	–	9 650	30 000	3,0	PCM 101212 B
	12	12	3	14 300	30 000	2,3	–
	12	15	–	12 000	37 500	3,8	PCM 101215 B
	12	15	3	18 000	37 500	2,9	–
	12	20	–	16 000	50 000	5,1	PCM 101220 B
	12	20	3	24 000	50 000	3,9	–
12	14	8	–	7 650	24 000	2,4	PCM 121408 B
	14	10	–	9 650	30 000	3,0	PCM 121410 B
	14	10	3	14 300	30 000	2,3	–
	14	12	–	11 600	36 000	3,6	PCM 121412 B
	14	12	3	17 300	36 000	2,8	–
	14	15	–	14 300	45 000	4,5	PCM 121415 B
	14	15	3	21 600	45 000	3,5	–
	14	20	–	19 300	60 000	6,0	PCM 121420 B
	14	20	3	29 000	60 000	4,6	–
	14	25	–	24 000	75 000	7,6	PCM 121425 B
	14	25	3	36 000	75 000	5,8	–
							PCM 121425 M

Composite bushings with metric dimensions

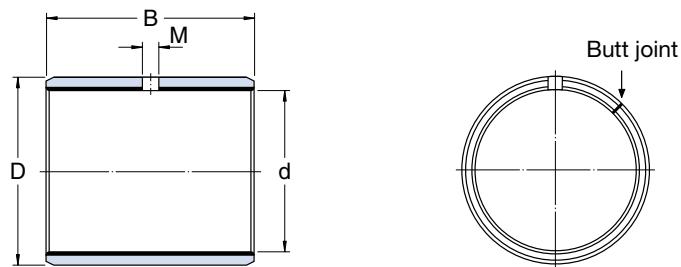
d 13 – 32 mm



Dimensions				Basic load ratings dynamic static		Mass	Designations	
d	D	B	M	C	C_0		PTFE composite	POM composite
mm				N		g	–	
13	15	10	–	10 400	32 500	3,2	PCM 131510 B	–
	15	10	3	15 600	32 500	2,4	–	PCM 131510 M
	15	20	–	20 800	65 500	6,3	PCM 131520 B	–
14	16	10	–	11 200	34 500	3,5	PCM 141610 B	–
	16	12	–	13 400	41 500	4,2	PCM 141612 B	–
	16	15	–	16 600	52 000	5,2	PCM 141615 B	–
	16	15	3	25 000	52 000	4,0	–	PCM 141615 M
	16	20	–	22 400	70 000	7,0	PCM 141620 B	–
	16	20	3	33 500	70 000	5,3	–	PCM 141620 M
	16	25	–	28 000	88 000	8,7	PCM 141625 B	–
	16	25	3	41 500	88 000	6,6	–	PCM 141625 M
15	17	10	–	12 000	37 500	3,7	PCM 151710 B	–
	17	10	3	18 000	37 500	2,8	–	PCM 151710 M
	17	12	–	14 300	45 000	4,4	PCM 151712 B	–
	17	12	3	21 600	45 000	3,4	–	PCM 151712 M
	17	15	–	18 000	56 000	5,6	PCM 151715 B	–
	17	15	3	27 000	56 000	4,3	–	PCM 151715 M
	17	20	–	24 000	75 000	7,4	PCM 151720 B	–
	17	25	–	30 000	93 000	9,3	PCM 151725 B	–
16	18	10	–	12 900	40 000	3,9	PCM 161810 B	–
	18	12	–	15 300	48 000	4,7	PCM 161812 B	–
	18	15	–	19 300	60 000	5,9	PCM 161815 B	–
	18	15	3	29 000	60 000	4,5	–	PCM 161815 M
	18	20	–	25 500	80 000	7,9	PCM 161820 B	–
	18	20	3	38 000	80 000	6,0	–	PCM 161820 M
	18	25	–	32 000	100 000	9,9	PCM 161825 B	–
	18	25	3	48 000	100 000	7,5	–	PCM 161825 M
18	20	15	–	21 600	67 000	6,6	PCM 182015 B	–
	20	15	3	32 500	67 000	5,0	–	PCM 182015 M
	20	20	–	29 000	90 000	8,8	PCM 182020 B	–
	20	20	3	43 000	90 000	6,7	–	PCM 182020 M
	20	25	–	36 000	112 000	11	PCM 182025 B	–
	20	25	3	54 000	112 000	8,4	–	PCM 182025 M

Composite bushings with metric dimensions

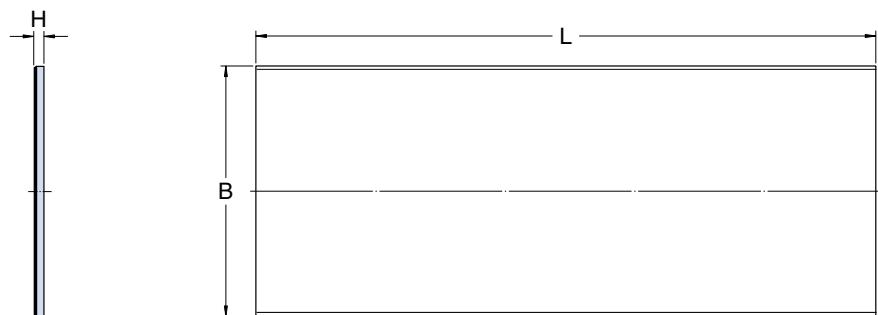
d 35 – 95 mm



Dimensions				Basic load ratings		Mass	Designations	POM composite
d	D	B	M	dynamic	static			
mm				N		g	–	
				C	C ₀			
35	39	20	—	54 000	166 000	34	PCM 353920 B	—
	39	20	4	80 000	166 000	31	—	PCM 353920 M
	39	30	—	81 500	255 000	52	PCM 353930 B	—
	39	30	4	122 000	255 000	47	—	PCM 353930 M
	39	40	—	110 000	345 000	68	PCM 353940 B	—
	39	50	—	137 000	430 000	87	PCM 353950 B	—
	39	50	4	208 000	430 000	78	—	PCM 353950 M
37	40	20	—	57 000	176 000	27	PCM 374020 B	—
	40	20	4	85 000	176 000	23	—	PCM 374020 M
	40	30	4	129 000	270 000	35	—	PCM 374030 M
40	44	20	—	61 000	193 000	39	PCM 404420 B	—
	44	20	4	91 500	193 000	36	—	PCM 404420 M
	44	30	—	93 000	290 000	59	PCM 404430 B	—
	44	30	4	140 000	290 000	53	—	PCM 404430 M
	44	40	—	125 000	390 000	78	PCM 404440 B	—
	44	40	4	190 000	390 000	66	—	PCM 404440 M
	44	50	—	156 000	490 000	98	PCM 404450 B	—
	44	50	4	236 000	490 000	89	—	PCM 404450 M
45	50	20	—	69 500	216 000	65	PCM 455020 B	—
	50	20	5	104 000	216 000	52	—	PCM 455020 M
	50	30	—	106 000	325 000	83	PCM 455030 B	—
	50	30	5	156 000	325 000	78	—	PCM 455030 M
	50	40	—	140 000	440 000	110	PCM 455040 B	—
	50	40	5	212 000	440 000	105	—	PCM 455040 M
	50	50	—	176 000	550 000	140	PCM 455050 B	—
	50	50	5	265 000	550 000	130	—	PCM 455050 M
50	55	20	—	76 500	240 000	62	PCM 505520 B	—
	55	30	—	116 000	365 000	93	PCM 505530 B	—
	55	30	5	176 000	365 000	86	—	PCM 505530 M
	55	40	—	156 000	490 000	125	PCM 505540 B	—
	55	40	5	236 000	490 000	115	—	PCM 505540 M
	55	60	—	236 000	735 000	185	PCM 505560 B	—
	55	60	5	355 000	735 000	170	—	PCM 505560 M

Dimensions							Basic load ratings dynamic	static	Mass	Designations	
d	D		B	M	C	C ₀				PTFE composite	POM composite
inch	mm	inch	mm	inch	mm	mm	N	g	–		
3,5	88,9	3,6875	93,663	2,5	63,5	–	440 000	1 370 000	330	PCZ 5640 B	–
				2,5	63,5	6	670 000	1 370 000	300	PCZ 5640 M	–
				3	76,2	–	530 000	1 660 000	395	PCZ 5648 B	–
				3	76,2	6	800 000	1 660 000	360	PCZ 5648 M	–
				3,75	95,25	–	670 000	2 080 000	495	PCZ 5660 B	–
				3,75	95,25	6	1 000 000	2 080 000	450	PCZ 5660 M	–
4	101,6	4,1875	106,363	3	76,2	–	610 000	1 900 000	450	PCZ 6448 B	–
				3	76,2	8	915 000	1 900 000	410	PCZ 6448 M	–
				3,75	95,25	–	765 000	2 400 000	565	PCZ 6460 B	–
				3,75	95,25	8	1 140 000	2 400 000	510	PCZ 6460 M	–
				4,75	120,65	–	965 000	3 050 000	715	PCZ 6476 B	–
				4,75	120,65	8	1 460 000	3 050 000	645	PCZ 6476 M	–
5	127	5,1875	131,763	3	76,2	–	765 000	2 400 000	560	PCZ 8048 B	–
				3,75	95,25	–	950 000	3 000 000	700	PCZ 8060 B	–
6	152,4	6,1875	157,163	3	76,2	–	915 000	2 850 000	670	PCZ 9648 B	–
				3,75	95,25	–	1 146 000	3 600 000	840	PCZ 9660 B	–
7	177,8	7,1875	182,563	3,75	95,25	–	1 340 000	4 150 000	975	PCZ 11260 B	–

Composite strip



Dimensions			Mass	Designations	
B	L	H	kg	PTFE composite	POM composite
<hr/>					
100	500	1,0	0,35	PCMS 1005001.0 B	–
	500	1,0	0,28	–	PCMS 1005001.0 M
100	500	1,5	0,55	PCMS 1005001.5 B	–
	500	1,5	0,46	–	PCMS 1005001.5 M
100	500	2,0	0,75	PCMS 1005002.0 B	–
	500	2,0	0,65	–	PCMS 1005002.0 M
100	500	2,5	0,55	PCMS 1005002.5 B	–
	500	2,5	0,85	–	PCMS 1005002.5 M

Other related products

Maintenance-free FW plain bearings

FW bushings are produced by a winding technique from a self-lubricating composite. The sliding layer consists of strands of high-strength polyester and reinforced PTFE (polytetrafluoroethylene) in an epoxy resin matrix. The shell or backing is made of wound high-strength tensioned glass fibre also in an epoxy resin matrix. The sliding and backing layers are firmly anchored to each other. Both layers are produced

by winding endless strands in a criss-cross pattern.

Modern filament winding technology has made it possible to combine the special mechanical properties of glass fibre with the excellent tribological properties of high-strength thermoplastic and PTFE fibres embedded in epoxy resin to produce a new innovative bearing material. The defined position of the strands in the criss-cross pattern and the intensive binding between the strands and the resin provide very high load carrying capacity and wear resistance.

The maintenance-free FW plain bearings are only available as cylindrical bushings. They are intended for radially loaded bearing arrangements for oscillating, rotational and linear movement, where there is risk of heavy edge loading and/or where chemical resistance under maintenance-free operation is required. They can be used at temperatures between -50 and +140 °C.

Assortment

The range of SKF maintenance-free FW bushings currently comprises bushings with bore diameters of 20 to 280 mm.

More information

More information about FW bushings can be found on CD-ROM 4700 "SKF Interactive Engineering Catalogue". Basic technical data are also given in leaflet Dd 7689 "Fibre composite instead of bronze: Maintenance-free FW bushings. The new plain bearing generation."



SKF spherical plain bearings

For arrangements where alignment movements have to be accommodated between two components in relative motion or where tilting movements or oscillations occur at relatively low sliding velocities, normally only spherical plain bearings are suitable. They have an inner ring with a spherically convex outside surface and an outer ring with a correspondingly spherically concave inside surface. Wherever spherical plain bearings are needed, whether maintenance can be provided or not, SKF spherical plain bearings will meet the bill.

The bearings requiring maintenance have the sliding contact surface combination steel-on-steel. As a rule, bearings with this sliding contact surface combination require regular relubrication. The high wear resistance of

the sliding surfaces makes these bearings especially suitable for arrangements where heavy loads of alternating direction, shock loads or heavy static loads have to be accommodated.

The SKF maintenance-free spherical plain bearings incorporate special sliding layers of advanced materials which have very low friction. They are used for applications where long bearing lives are required without maintenance, or where operating conditions like inadequate or total absence of lubrication do not allow the use of steel-on-steel bearings. SKF maintenance-free bearings are produced with three different sliding contact surface combinations, depending on bearing size and series. These three sliding contact surface combinations are

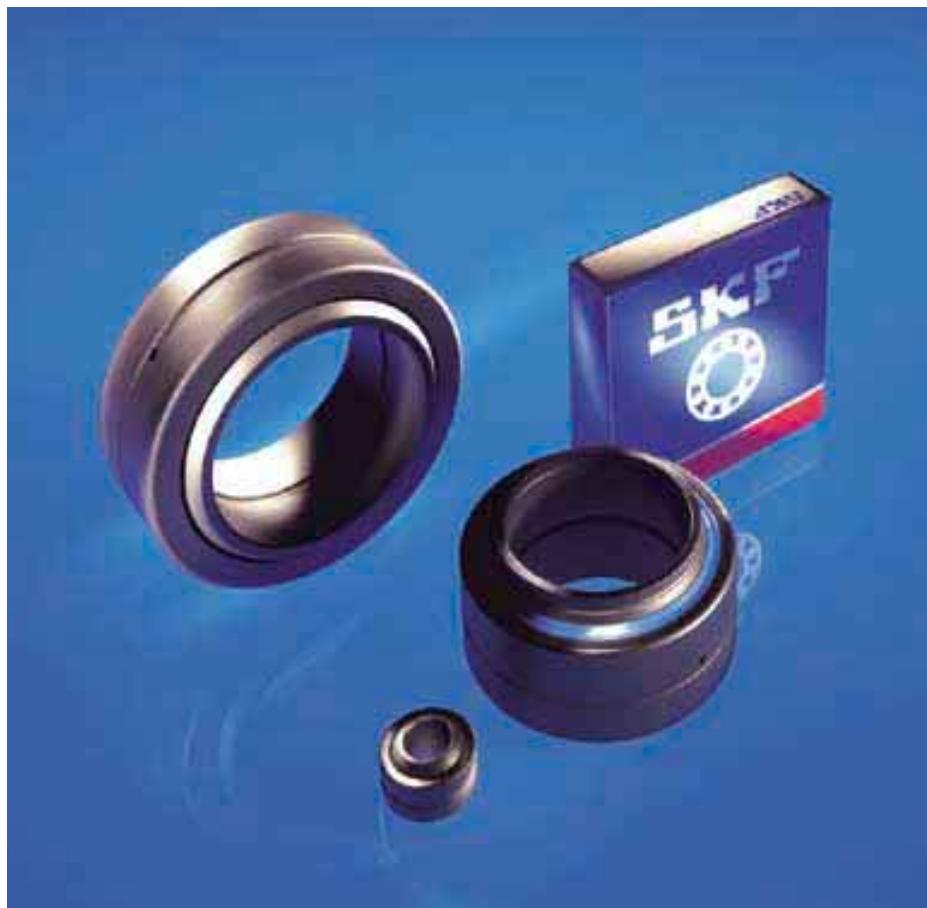
- steel/sinter bronze composite,
- steel/PTFE fabric, and
- steel/PTFE composite.

Assortment

The comprehensive SKF assortment comprises radial, angular contact and thrust bearings and covers a journal diameter range between 4 and 1 250 mm.

Further information

For detailed information about SKF spherical plain bearings please refer to CD-ROM 4700 "SKF Interactive Engineering Catalogue" or brochure 4407 "SKF spherical plain bearings and rod ends".



SKF rod ends

Rod ends have been designed for use in construction and control rod linkages, for the end of a piston rod or the base of pneumatic or hydraulic cylinders where tractive and compressive loads have to be transmitted at the same time as alignment movements have to be accommodated. A rod end is basically an eye-shaped head with integral shank (the housing) and a self-aligning bearing. Most SKF rod ends incorporate a standard spherical plain bearing which is held in the housing bore by staking at both sides or by retaining rings. The wide range of SKF rod ends contains products suitable for heavy alternating loads as well as for operation without maintenance.

The SKF rod ends requiring maintenance are available with the sliding contact surface combinations steel-on-steel and steel-on-bronze, the

majority being steel-on-steel rod ends. These rod ends generally require periodic relubrication. The high wear resistance of the sliding surfaces makes steel-on-steel rod ends especially suitable for arrangements where loads of alternating direction or relatively heavy static loads have to be accommodated. Under such operating conditions but where lubricant starvation may be encountered, steel-on-bronze rod ends are recommended.

SKF maintenance-free rod ends are produced with three different sliding contact surface combinations, depending on size and series. These three sliding contact surface combinations are

- steel/sinter bronze composite,
- steel/PTFE fabric, and
- steel/PTFE composite.

The sliding layers of modern materials, which have very low friction, are

the same as those used for SKF maintenance-free spherical plain bearings.

Assortment

The SKF assortment comprises rod ends with female or male thread (left- or right-hand) and also with welding shank. The latter are only available as steel-on-steel rod ends as standard. The assortment covers a pin diameter range between 5 and 200 mm.

Further information

For detailed information about SKF rod ends please refer to CD-ROM 4700 "SKF Interactive Engineering Catalogue" or brochure 4407 "SKF spherical plain bearings and rod ends".



Application examples

The unique properties and excellent performance of dry sliding bearings have led to their use in a variety of industrial, domestic and other applications. They are particularly suitable for conditions where both maintenance and lubrication are either not required or not possible. Examples of typical existing applications for these bearings are as follows.

Automotive

King pins, starter pinions, brake rod linkages, brake shafts, brake shoes, suspensions, window lifts, foot pedals, accelerator linkages, fans, propeller shafts, clutch release levers, steering rods, steering columns, swinging arms, shock absorbers, carburettor butterfly valves etc.

Rail vehicles, railway installations

Automatic doors, level crossing barriers, brakes, pantographs, controllers, load switches, relay boxes, signalling equipment, wagons, points etc.

Aerospace

Brakes, electronic equipment, undercarriages, engines, radar equipment, control devices etc.

Construction industry, conveying equipment

Lifts, excavator drives, excavator arms, excavator control equipment, concrete mixers, fork lift trucks, hydraulic rams, chain tensioning sprockets, crane drives, crane control equipment, crane jibs, mortar carriers, pallet lift trucks, pneumatic lifts, caterpillar graders, escalators, moving pavements, vibrating screens, slides, shuttering cleaning

machines, low loader trailers, winches, conveyors of all kinds etc.

Office machines and equipment

Addressing machines, data processing equipment, tape recorders, swivelling chairs, franking machines, copiers, blueprint copiers, drawing tables, drawing machines etc.

Domestic appliances, hospital equipment

Dental equipment, dishwashers, ironing machines, air conditioning equipment, hospital beds, refrigerators, sewing machines, operating tables, X-ray equipment, vacuum cleaners, washing machines etc.

Machines for agriculture and food-stuff industries

Bottling machines, bakery equipment, timber saws, filtering centrifuges, abattoir and meat processing equipment, hay tedders, potato harvesters, wine-making equipment, loaders, unloaders, combine harvesters, mills, planting machines, root crop harvesters, balers, tractors, tractor seats, automatic packaging equipment, weighing equipment etc.

General engineering

Chamfering machines, bending machines, machines for sheet metal working, briquetting machines, forge machinery, woodworking machines, plastic moulding machines, presses, automatic welding equipment, machine tools, crushing plant etc.

Papermaking and textiles

Cutting machines, printing machines, doubling machines, folding machines, yarn and wool machines, vulcanising machines, carders, button machines, papermaking and paper-treating machinery, sorting devices, spinning machines, stuffers, knitting machines, looms etc.

Pumps, valves

Axial and radial piston pumps, metering pumps, firefighting pumps, compressors, ball cocks, mixer valves, oil burners, pumps for chemicals, regulating valves, submersible pumps, vacuum pumps, spur gear pumps etc.

Electrical equipment

Starting levers for electric motors, contactors, control equipment, switch gear etc.

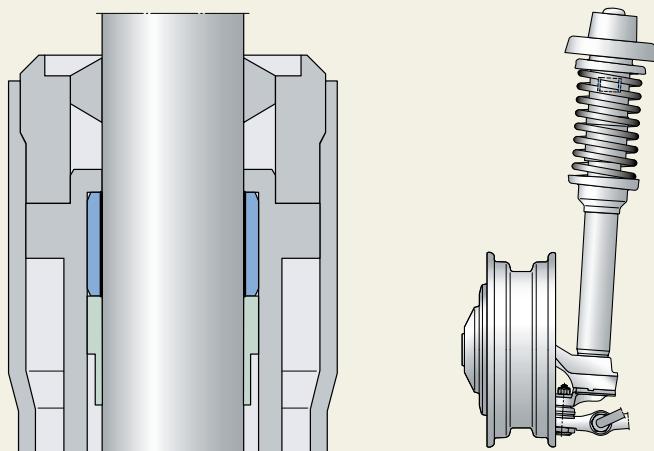
Automatic devices, tools

Charging and feeding devices, vending machines, pneumatic tools, hydraulic tools etc.

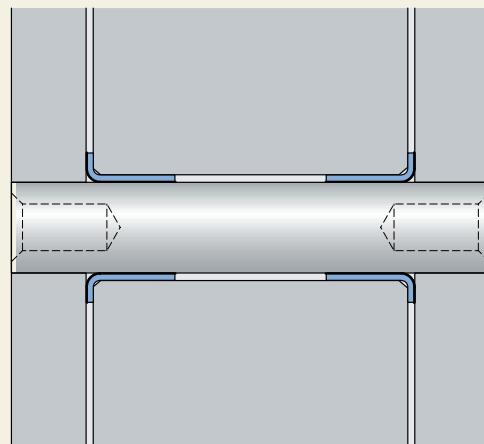
Other uses

Refuse disposal equipment and plant, brake magnets, heat treatment plant, blinds, awnings, smelting furnaces, continuously variable gears, drying plant, steel construction etc.

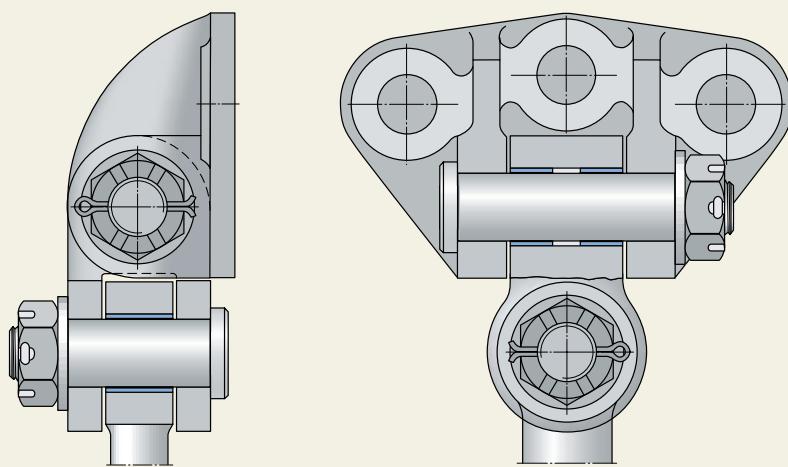
Linear guide for a car suspension strut piston rod with a PTFE composite bushing



Bearing arrangement for a sunshade (awning) linkage with flanged PTFE composite bushings

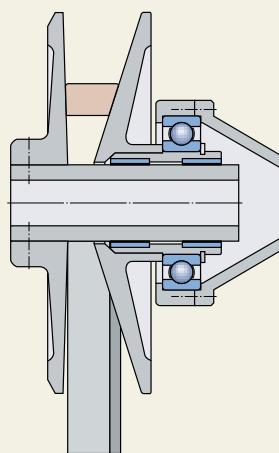


Bearing arrangement for the leaf spring attachment to the upper linkage of a bogie with PTFE composite bushings

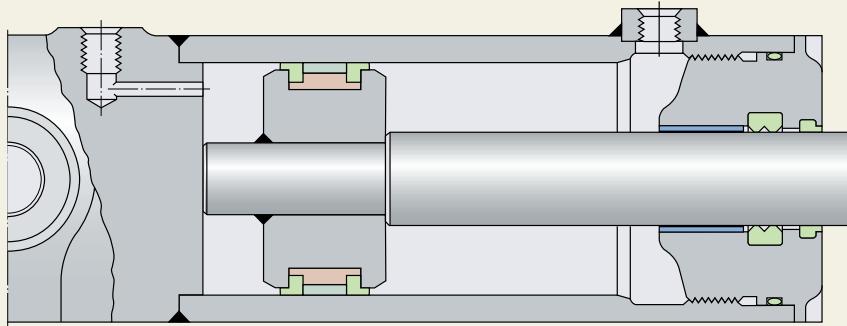


Application examples

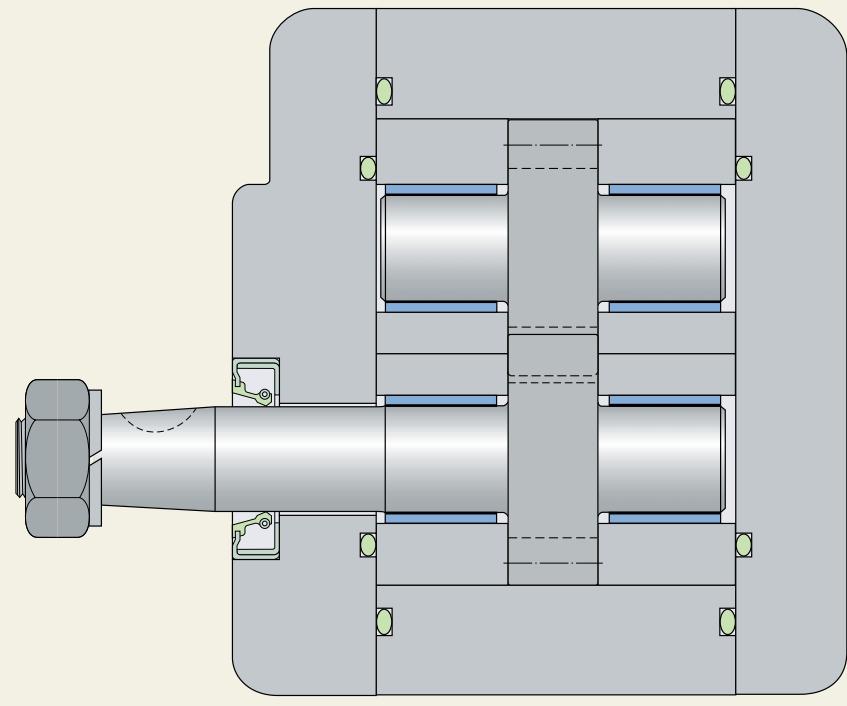
**Linear guide for a continuously variable
governor gear pulley with POM composite
bushings**



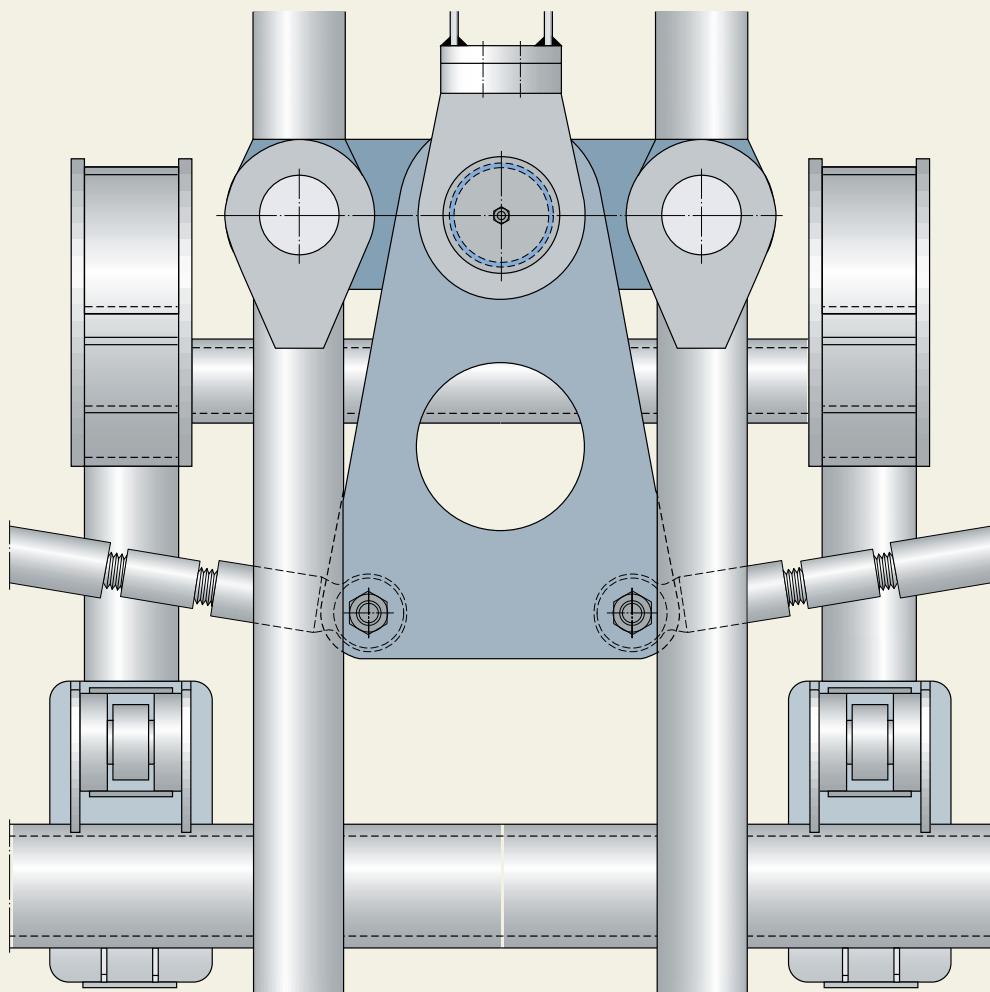
**Linear guide for a hydraulic cylinder
piston rod with a POM composite
bushing**



**Bearing arrangement for a geared pump
with PTFE composite bushings**

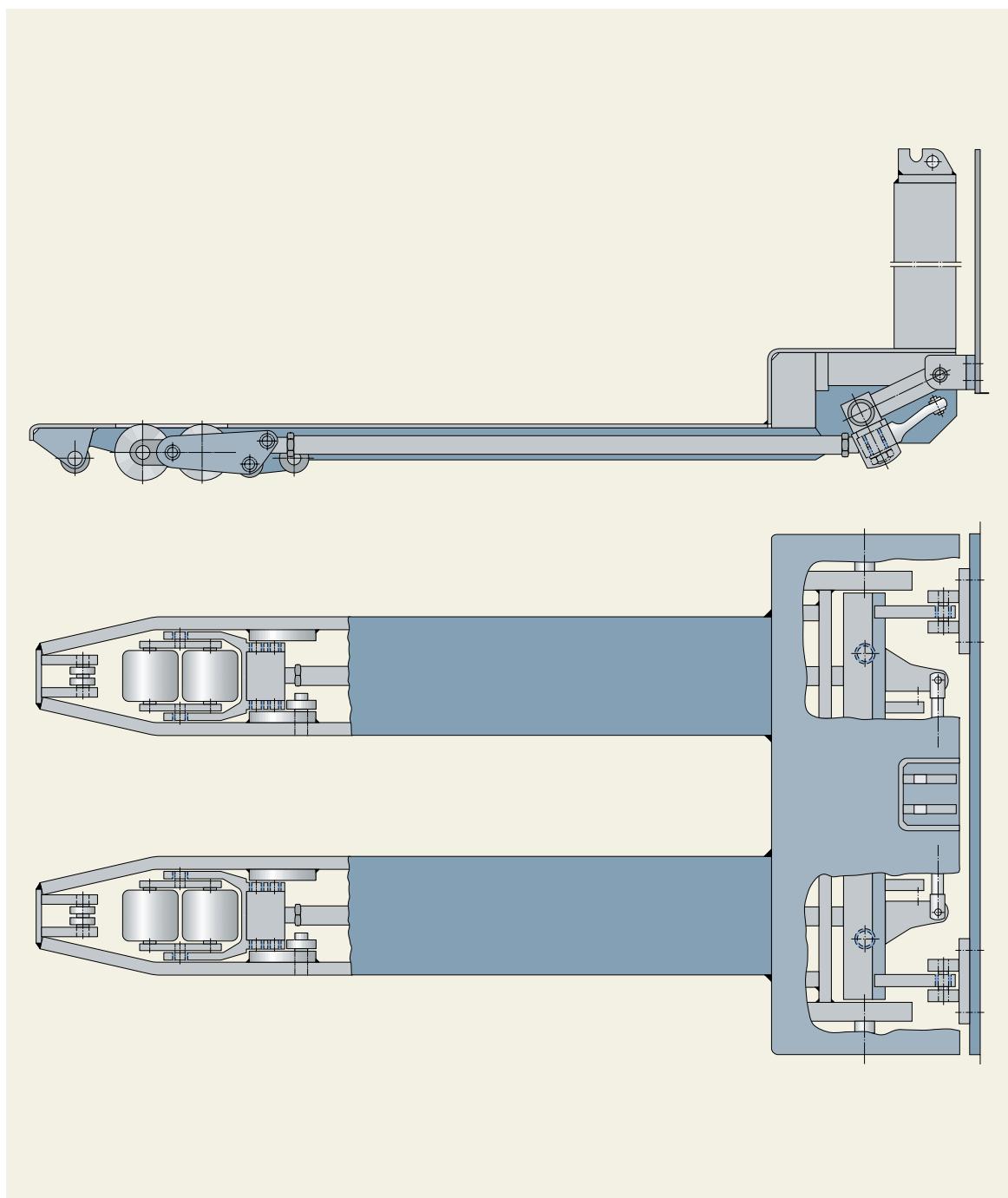


**Bearing arrangement
for a heavy
loader trailer
steering gear shift
lever with POM
composite bush-
ings**

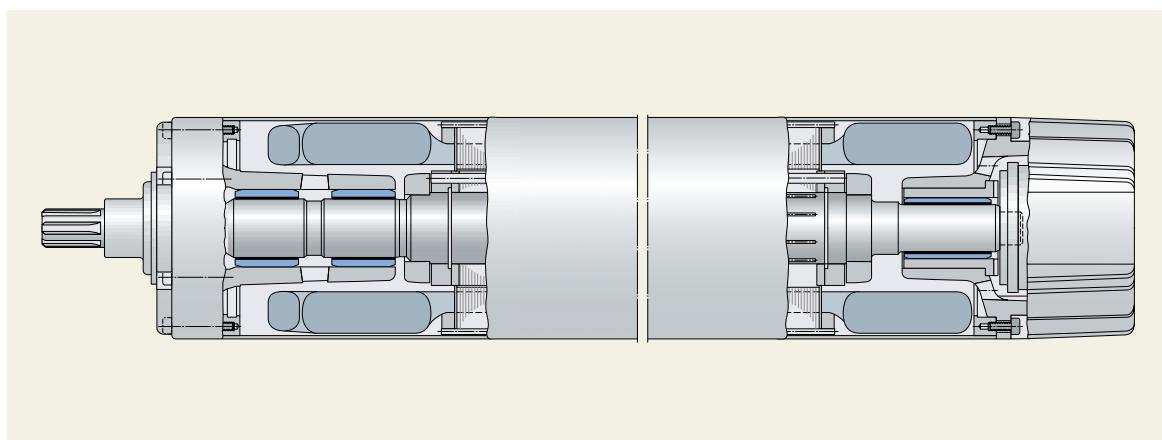


Application examples

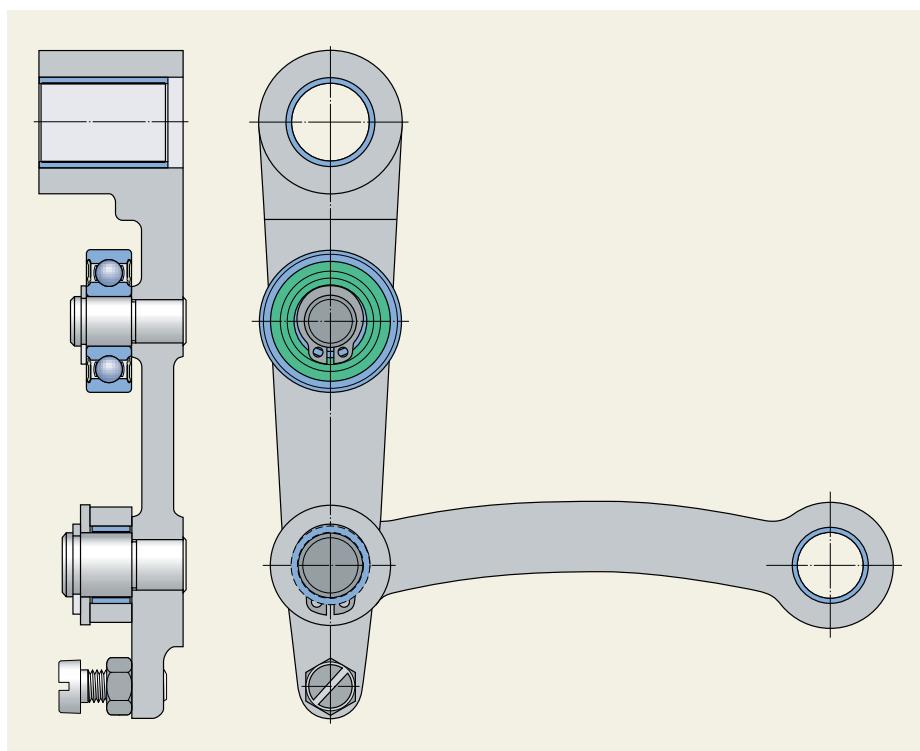
Bearing arrangement for a pallet lift truck loading frame coupling rod with POM composite bushings



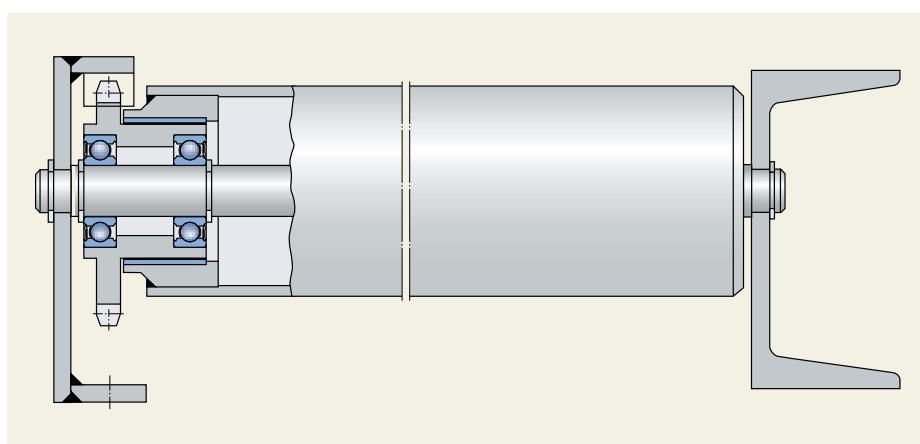
Bearing arrangement for the rotor on an underwater pump with PTFE composite bushings



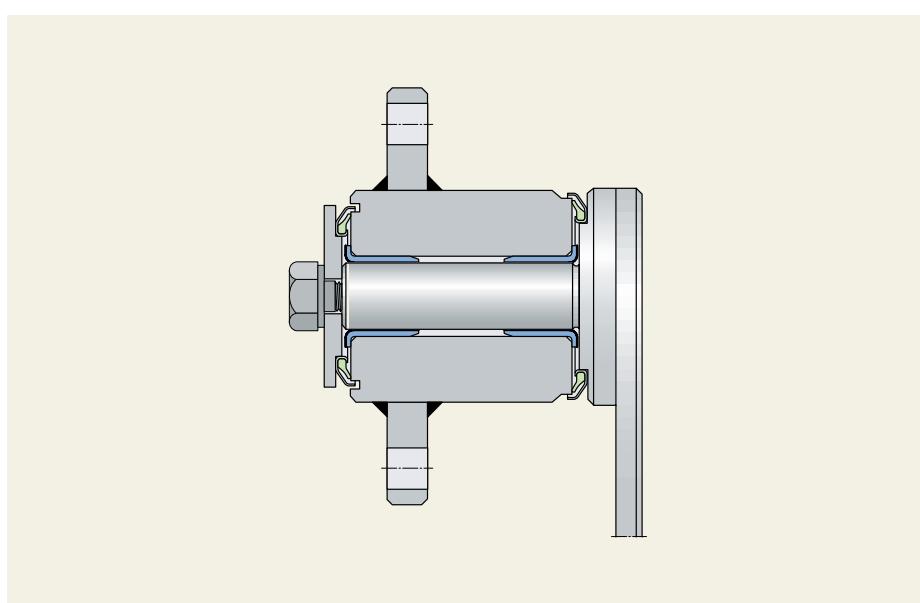
Coupling rod bearing arrangements for a printing press gear lever with PTFE composite bushings



Bearing arrangement for a conveyor roller with a PTFE composite bushing

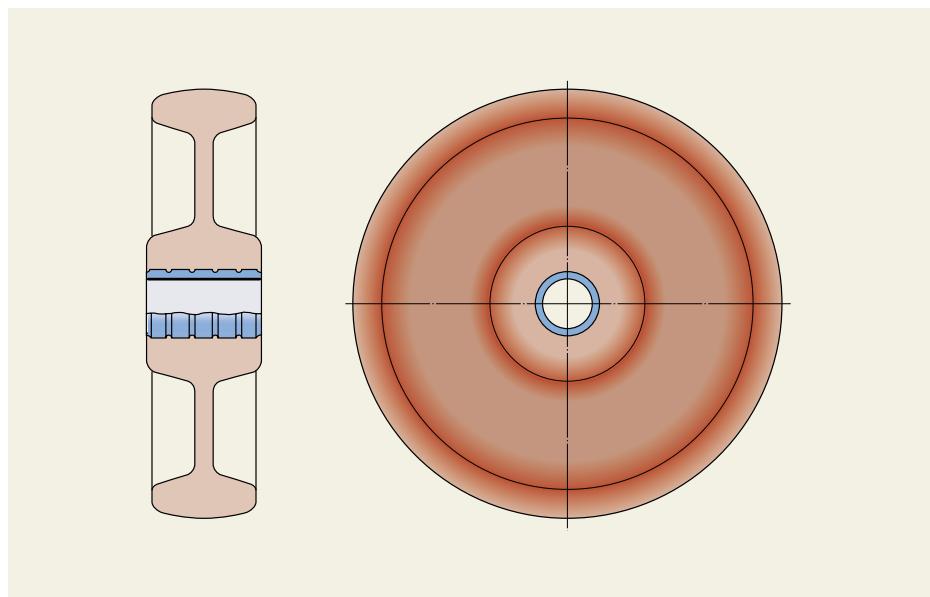


Bearing arrangement for the cutter on debarking equipment with flanged PTFE composite bushings

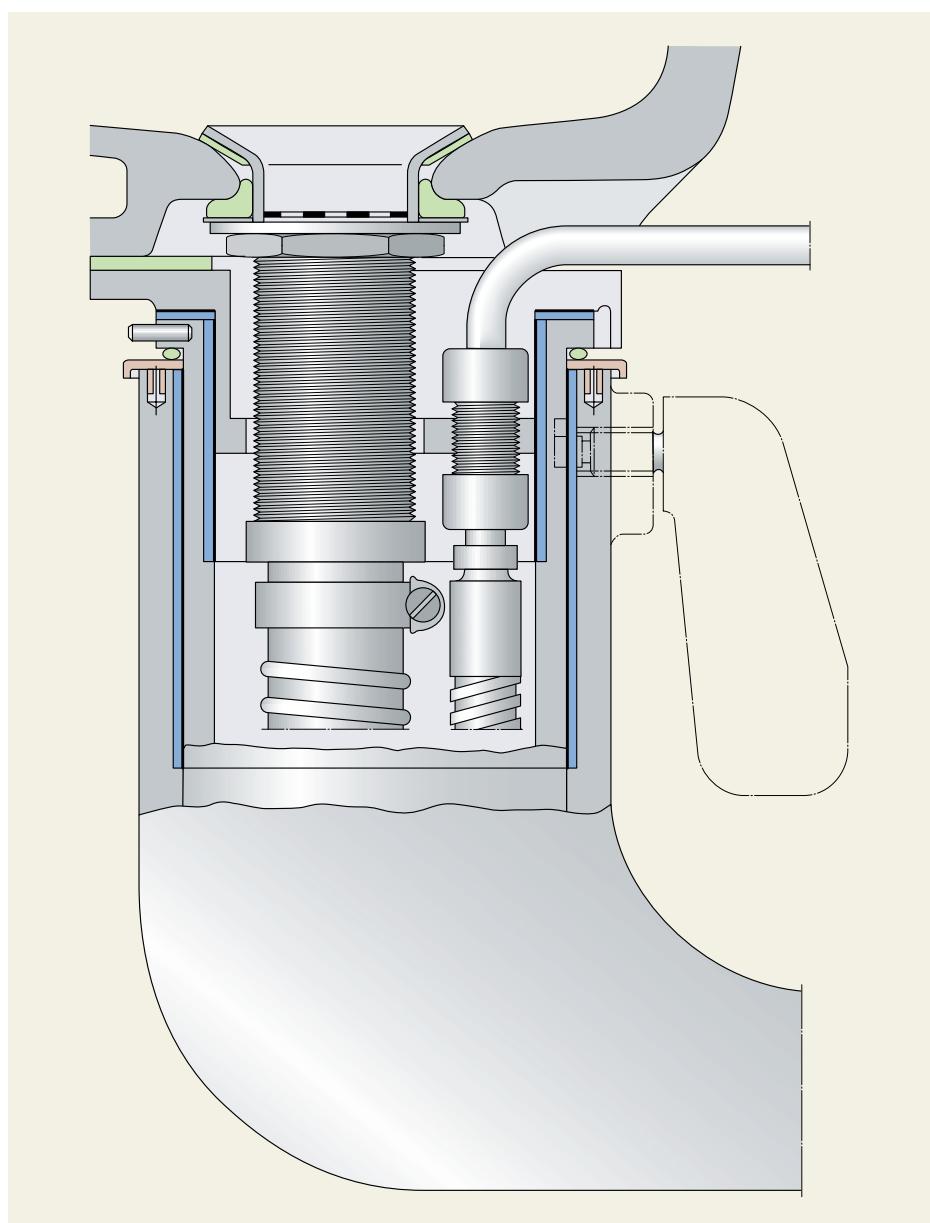


Application examples

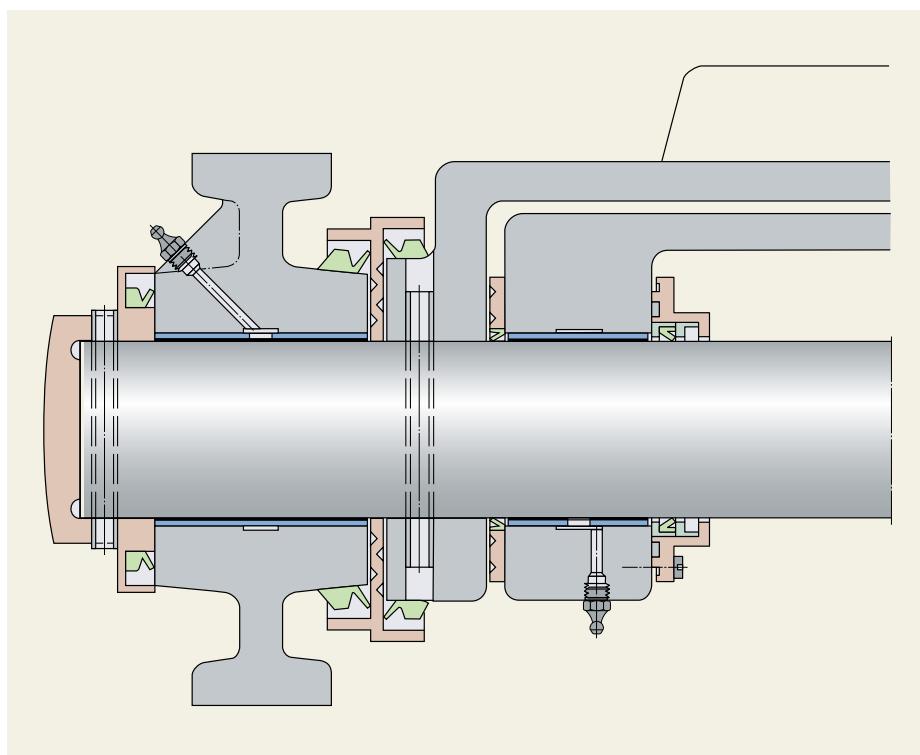
Bearing arrangement for the castors on an airline catering trolley with PTFE composite bushings



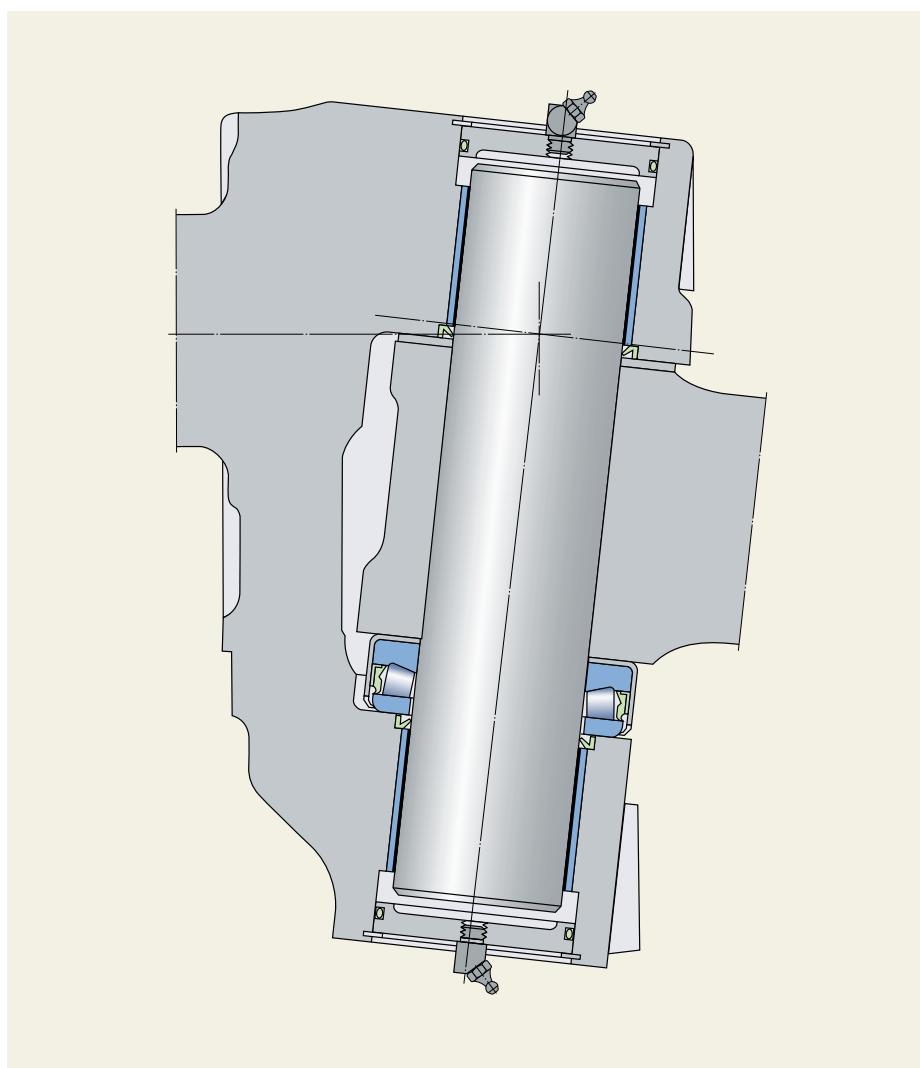
Swivelling device for a wash basin column with POM composite bushings



Bearing arrangement for the platform and side rollers of a platform conveyor with POM composite bushings

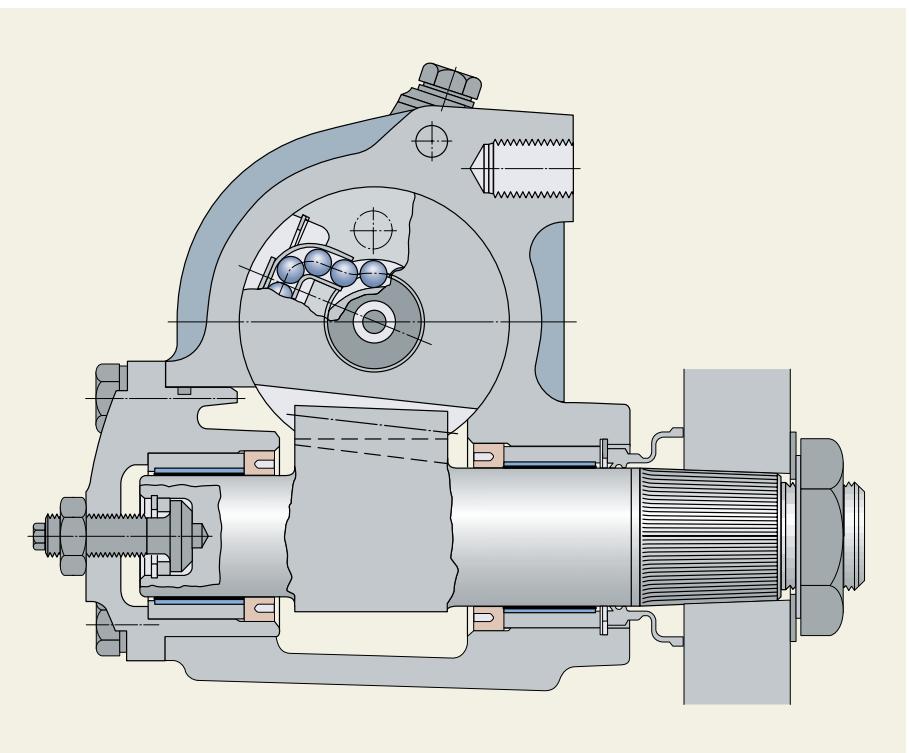


King pin bearing arrangement with POM composite bushings

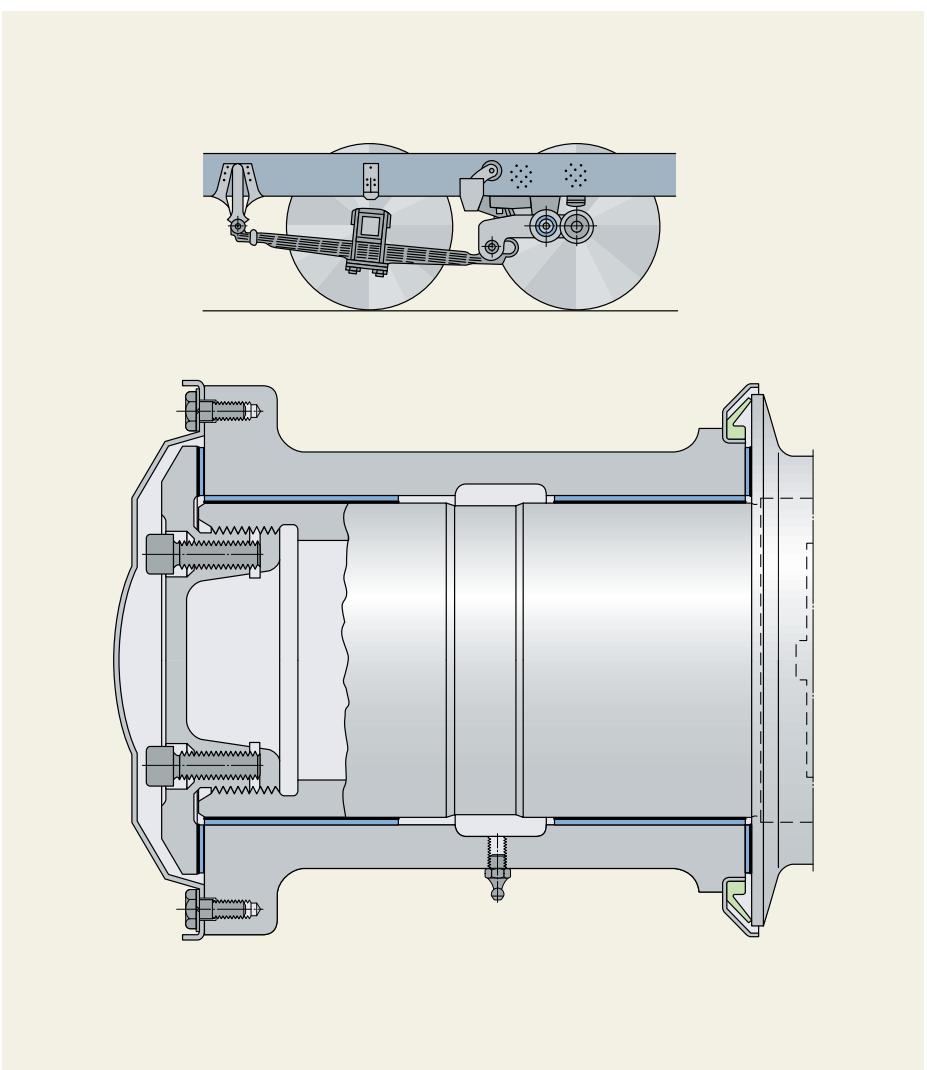


Application examples

**Linear guide for a truck servo steering
with PTFE composite bushings**



**Bearing arrangement for a lift-off axle
with POM composite bushings and
thrust washers**



The SKF group – a worldwide corporation

SKF is an international industrial Group operating in some 130 countries and is world leader in bearings.

The company was founded in 1907 following the invention of the self-aligning ball bearing by Sven Wingquist and, after only a few years, SKF began to expand all over the world.

Today, SKF has some 45 000 employees and around 80 manufacturing facilities spread throughout the world. An international sales network includes a large number of sales companies and some 7 000 distributors and retailers. Worldwide availability of SKF products is supported by a comprehensive technical advisory service.

The key to success has been a consistent emphasis on maintaining the highest quality of its products and services. Continuous investment in research and

development has also played a vital role, resulting in many examples of epoch-making innovations.

The business of the Group consists of bearings, seals, special steel and a comprehensive range of other high-tech industrial components. The experience gained in these various fields provides SKF with the essential knowledge and expertise required in order to provide the customers with the most advanced engineering products and efficient service.





The SKF Group is the first major bearing manufacturer to have been granted approval according to ISO 14001, the international standard for environmental management systems. The certificate is the most comprehensive of its kind and covers more than 60 SKF production units in 17 countries.



The SKF Engineering & Research Centre is situated just outside Utrecht in The Netherlands. In an area of 17 000 square metres (185 000 sq.ft) some 150 scientists, engineers and support staff are engaged in the further improvement of bearing performance. They are developing technologies aimed at achieving better materials, better designs, better lubricants and better seals – together leading to an even better understanding of the operation of a bearing in its application. This is also where the SKF Life Theory was evolved, enabling the design of bearings which are even more compact and offer even longer operational life.



SKF has developed the Channel concept in factories all over the world. This drastically reduces the lead time from raw material to end product as well as work in progress and finished goods in stock. The concept enables faster and smoother information flow, eliminates bottlenecks and bypasses unnecessary steps in production. The Channel team members have the knowledge and commitment needed to share the responsibility for fulfilling objectives in areas such as quality, delivery time, production flow etc.



SKF manufactures ball bearings, roller bearings and plain bearings. The smallest are just a few millimetres (a fraction of an inch) in diameter, the largest several metres. SKF also manufactures bearing and oil seals which prevent dirt from entering and lubricant from leaking out. SKF's subsidiaries CR and RFT S.p.A. are among the world's largest producers of seals.



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Publication 4413 E

Printed in Sweden on environmentally friendly, chlorine-free paper (Multiart Silk) by SG idé & tryck ab.